

Evolution of zero-pressure-gradient boundary layers from different tripping conditions

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In this paper we study the spatial evolution of zero-pressure-gradient (ZPG) turbulent boundary layers from their origin to a canonical high-Reynolds-number state. A prime motivation is to better understand under what conditions reliable scaling behaviour comparisons can be made between different experimental studies at matched local Reynolds numbers. This is achieved here through detailed streamwise velocity measurements using hot wires in the large University of Melbourne wind tunnel. By keeping the unit Reynolds number constant, the flow conditioning, contraction and trip can be considered unaltered for a given boundary layer's development and hence its evolution can be studied in isolation from the influence of inflow conditions by moving to different streamwise locations. Careful attention was given to the experimental design in order to make comparisons between flows with three different trips while keeping all other parameters nominally constant, including keeping the measurement sensor size nominally fixed in viscous wall units. The three trips consist of a standard trip and two deliberately 'over-tripped' cases, where the initial boundary layers are over-stimulated with additional large-scale energy. Comparisons of the mean flow, normal Reynolds stress, spectra and higher-order turbulence statistics reveal that the effects of the trip are seen to be significant, with the remnants of the 'over-tripped' conditions persisting at least until streamwise stations corresponding to $Re_x = 1.7 \times 10^7$ and $x = O(2000)$ trip heights are reached (which is specific to the trips used here), at which position the non-canonical boundary layers exhibit a weak memory of their initial conditions at the largest scales $O(10\delta)$, where δ is the boundary layer thickness. At closer streamwise stations, no one-to-one correspondence is observed between the local Reynolds numbers (Re_τ , Re_θ or Re_x etc.), and these differences are likely to be the cause of disparities between previous studies where a given Reynolds number is matched but without account of the trip conditions and the actual evolution of the boundary layer. In previous literature such variations have commonly been referred to as low-Reynolds-number effects, while here we show that it is more likely that these differences are due to an evolution effect resulting from the initial conditions set up by the trip and/or the initial inflow conditions. Generally, the mean velocity profiles were found to approach a constant wake parameter Π as the three boundary layers developed along the test section, and agreement of the mean flow parameters was found to coincide with the location where other statistics also converged, including higher-order moments up to tenth order. This result therefore implies that it may be sufficient to document the mean flow parameters alone in order

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to ascertain whether the ZPG flow, as described by the streamwise velocity statistics, has reached a canonical state, and a computational approach is outlined to do this. The computational scheme is shown to agree well with available experimental data.

Key words: turbulent boundary layers, turbulent flows

1. Introduction

In the study of different wall-bounded turbulent flows, the zero-pressure-gradient (ZPG) turbulent boundary layer is probably the most investigated, and perhaps the most reviewed. The ZPG boundary layer is also often considered as the baseline flow when examining various external influences such as roughness, streamwise pressure gradient, heat transfer, etc. However, significant challenges still exist in identifying the true statistical behaviour of a ZPG boundary layer, particularly at high Reynolds numbers (Marusic *et al.* 2010*b*). One of the major challenges lies in unravelling under what conditions different experimental results can be credibly compared when assessing various scaling laws, such as outer-flow similarity. Comparisons of previous experimental data sets have shown significant discrepancies between different studies even though local experimental parameters are matched (Chauhan & Nagib 2008; Chauhan, Nagib & Monkewitz 2009). Conventional wisdom suggests that provided the local Reynolds number is matched then comparisons can be made for assessing scaling behaviour, and this implicitly implies that no consideration is given to the evolution of the boundary layers. In the following we will show that such comparisons are not valid under certain conditions. Moreover, when anomalous behaviours have been noted in the literature, for example, such as the variation of Coles wake factor for $Re_{\delta^*} < 5000$ (Coles 1962), these have been described as a low-Reynolds-number effect. Again, in the following we will show that such trends are better described as an evolution effect resulting from the initial conditions set up by the trip and/or the initial inflow conditions.

Anomalous discrepancies between scaled data are not confined to experimental studies. Schlatter & Örlü (2010) revealed large differences between direct numerical simulations (DNS) of the ZPG boundary layer for the shape factor $H = \delta^*/\theta$ and skin-friction coefficient $C_f = 2(U_\tau/U_\infty)^2$ (δ^* is the displacement thickness, θ is the momentum thickness, U_τ is the skin-friction velocity and U_∞ is the free-stream velocity). A likely cause for these disparities is the different evolution conditions for each boundary layer. A systematic DNS study by Schlatter & Örlü (2012) re-simulated the canonical ZPG flow with varying inflow conditions and tripping effects. Schlatter & Örlü (2012) suggest that the mean statistics and integral parameters agree well in the inner and outer layer for $Re_\theta > 2000$, provided transition is initiated inside the boundary layer at a low enough Re_θ (< 300 , where $Re_\theta = U_\infty\theta/\nu$ is the Reynolds number based on free-stream velocity U_∞ , momentum thickness θ and kinematic viscosity ν), and that under- or over-stimulation by the trip is avoided. They also concluded that the outer region of the boundary layer requires a longer inflow length to reach a fully turbulent state. Since direct numerical simulations are still at the lower end of the Reynolds number range where a scale separation starts to establish itself, currently the only way to investigate the influence of different tripping or inflow conditions in medium- to high-Reynolds-number boundary layers is through experiments. This is the approach used in the present study where we experimentally study the evolution of high-Reynolds-number ZPG turbulent boundary layers from different initial (tripping) conditions.

Although researchers have long recognised the importance of initial conditions in boundary layers, quantifiable and careful investigations of the role of the upstream conditions on the downstream flow have been sparsely reported. One of the first experiments performed in this area was by Erm & Joubert (1991) who investigated the effect of various tripping conditions on turbulent boundary layers for Reynolds numbers between $715 \leq Re_\theta \leq 2810$. Erm & Joubert (1991) proposed a technique for obtaining correctly stimulated turbulent boundary layers for a particular tripping device by changing the dimension of the trip iteratively until the measured Coles (1956) wake factor Π agrees with the Coles (1962) curve of Π versus Re_θ . Here $2\Pi/\kappa$ is the maximum deviation of the mean velocity profile from the log-law in the outer region. They also mentioned that the effectiveness of a given device in tripping a flow could be gauged to some extent by an examination of a plot of the associated C_f versus x relationship, where x is the streamwise distance from the trip. Thus it was reasoned that if a given device was subjected to a series of free-stream velocities and plots of C_f were produced for each velocity within the range, then an examination of the entire family of plots may indicate a velocity that will later, after further testing, be shown to be the velocity that leads to the Coles (1962) relationship being satisfied by the flow. This was validated by Erm & Joubert (1991) with measurements. They made it very clear that each free-stream velocity would require a unique tripping device to properly stimulate the turbulent boundary layers. When the velocity is changed from the design value of the trip, the resulting deviation from the log-law is known to depart from the ideal form suggested by Coles (1962). The degree to which this deviation changes, and also how the mean flow parameters vary for a given Reynolds number with changes in flow velocity for different tripping devices, is yet to be investigated. Researchers have often presented measurements corresponding to fixed tripping configurations, but different free-stream velocities, based on the assumption that it is only the value of the local Reynolds number (e.g. Re_θ) that is important and not how the boundary layer was formed, i.e. no record of boundary layer evolution is necessary (see discussion in Hutchins 2012). However, studies by Johansson & Castillo (2001), Castillo & Johansson (2002) and Castillo & Walker (2002) showed that the mean velocity and Reynolds stress profiles collapse only in the outer part of the boundary layer when scaled in outer variables, provided the upstream conditions were fixed between the flows, and they concluded that the variation in Reynolds numbers in the boundary layer is mainly due to changes in the initial conditions.

1.1. Mean flow considerations

The classical self-similarity theory states that the mean velocity deficit profiles $U_\infty^+ - U^+$ (U_∞ is the free-stream velocity) of all ZPG boundary layers would be self-similar in the outer region when the Reynolds number is sufficiently high. The velocity deficit is generally expressed as a two-parameter relation utilising $\eta = z/\delta$ (here z is the wall-normal position and δ is the boundary layer thickness) and the wake parameter Π (Coles 1956). The functional form of the velocity deficit profile is as follows:

$$\begin{aligned} \frac{U_\infty - U}{U_\tau} &= -\frac{1}{\kappa} \ln(\eta) + \frac{2\Pi}{\kappa} [1 - W(\eta, \Pi)] \\ &= f(\eta, \Pi). \end{aligned} \quad (1.1)$$

Here W is the wake profile such that $W(1, \Pi) = 1$ and it models the deviation of the mean velocity from the logarithmic law. Note that in this definition of Π , the

magnitude of Π represents the deviation of U^+ from the log-law at $y = \delta$ instead of the definition utilised by Coles (1956) where Π is the maximum deviation from the log-law. The velocity deficit relation postulates that the self-similarity solution does not depend on the local Reynolds number: in other words, there is no ‘ x ’ (streamwise distance) dependence when there is a sufficient boundary layer development length. The data surveyed over a wide range of Reynolds numbers by Coles (1962) showed that the wake parameter Π becomes nominally constant for $Re_\theta \geq 8000$. The scatter in the data at low Reynolds numbers not only led to questions about the validity of the experiments but also motivated new theories by Castillo & Johansson (2002), based on the ‘asymptotic invariance principle’ of George & Castillo (1997). The principle asserts that in the limit as $Re \rightarrow \infty$, the boundary layer equations become independent of the friction Reynolds number ($Re_\tau = \delta U_\tau / \nu$), but the effect of the upstream conditions might be retained because the flow may always depend on them, even in the asymptotic limit. According to Castillo & Johansson (2002), the velocity deficit (1.1) assumes the following form:

$$\frac{U_\infty - U}{U_\tau} = f(\eta, \Pi, *) \quad (1.2)$$

(with a similar form also existing with U_∞ as the velocity scale rather than U_τ), where ‘ $*$ ’ represents the upstream conditions and retains an influence over all Reynolds numbers, including the asymptotic high- Re case.

Another possible explanation for the scatter in the data used by Coles (1962) is the strong dependence of Π on the method by which it is extracted from the data and on how sensitive Π and δ are to the particular curve fit used for the outer part of the mean velocity profile. This dependence was studied extensively by Monkewitz, Chauhan & Nagib (2008). In recent times, the progress in quantifying the deviation from the canonical state of the boundary layers has been encouraging; however, more data from different facilities are required before many of the questions can be answered satisfactorily. Considerable advances have been made by Monkewitz, Chauhan & Nagib (2007), Nagib, Chauhan & Monkewitz (2007) and Chauhan *et al.* (2009), who have proposed criteria to quantify when ZPG boundary layers are well behaved, i.e. they are representative of the canonical state. In the canonical state, evolution of mean flow parameters (such as Π or H) is governed only by the self-similarity of the mean defect velocity profile in the inertially dominated region. Their criteria are based on the assumption that the canonical asymptotic state is attained when the wake parameter Π becomes invariant and/or the values of skin friction and shape factor are consistent with each other in the classical framework. Although it is satisfying that many high-quality experiments agree well with the asymptotic turbulent boundary layer theory and satisfy the criteria outlined by Chauhan *et al.* (2009), the prediction of development of boundary layers at low Re is challenging due to the sensitivity of the boundary layer to influences immediately upstream.

This difficult problem of how the canonical asymptotic state evolves from an arbitrary initial condition (or a low- Re state) was first tackled by Perry, Marusic & Li (1994) and Perry, Marusic & Jones (2002) who computed the development of the ZPG boundary layer from a specified set of initial conditions using the momentum and continuity equations in simplified form. This involves the hypothesis that the total shear stress field is uniquely described by a two-parameter family. In addition, a relation between the mean flow and shear stress parameters is required to close the

system of equations. Perry *et al.* (1994) and Perry *et al.* (2002) have tentatively given a closure equation based on the data sets available to them at that time and showed that all the different evolution curves converge to a single bifurcation line, which can be regarded as equivalent to the asymptotic curves proposed by Nagib *et al.* (2007) to indicate when an evolving boundary layer becomes well behaved.

1.2. Paper outline

The main aim of the current work is to obtain high-quality data of boundary layers that develop spatially to high Reynolds number from different initial conditions, and to ascertain whether the boundary layers ever reach a well-defined or ‘fully developed’ state, allowing for meaningful comparisons of scaling behaviours between different experiments. This is done by considering three tripping devices for the same upstream conditions in a purpose-built wind tunnel with a long (27 m) working section. In the remainder of the paper a detailed description of the experiments is given, followed by comparisons of the mean flow, second-order statistics, including spectra, and high-order statistics of the streamwise velocity across the boundary layers at multiple streamwise stations. The second main aim of the study is to see where the classical approach such as the one developed by Perry *et al.* (2002) can be used to compute the evolution of the flow in different developing states. This approach will be summarised in § 5 and computations and comparisons to experimental data will be presented in § 6.

2. Experimental setup

We here produce a set of detailed boundary layer measurements in a developing turbulent boundary layer at matched inlet conditions (achieved by matching the unit Reynolds number U_∞/ν between experiments). Matched unit Reynolds number allows measurements made at the same x location, yet acquired on different days, to have the same Re_x even if the ambient temperature is different due to daily variations. In this case variations in friction Reynolds number, $Re_\tau = U_\tau\delta/\nu$, are attained by measuring at increasing streamwise distances downstream of the tripped inlet to the working section (from $1.6 < x < 18$ m) rather than by altering U_∞ or ν . By keeping the unit Reynolds number constant, the flow conditioning, contraction, and trip can be considered unaltered for all measurements and hence the effect of streamwise development can be studied in isolation from the influence of inflow conditions. By contrast, in facilities where variation in Re_τ is attained by increasing the free-stream velocity (or reducing ν), one runs the risk of contaminating any genuine Reynolds number development with variations in the inflow conditions. For example, in such situations, even if the contraction, fan and flow conditioning perform similarly, and even if the pressure gradient is carefully adjusted, the magnitude of the tripping stimulation will increase with U_∞/ν (see Erm & Joubert 1991).

2.1. Facility

The measurements were performed in the High-Reynolds-Number Boundary Layer Wind Tunnel in the Walter Basset Aerodynamics Laboratory at the University of Melbourne. This tunnel has a 27 m long working section, and is capable of generating free-stream velocities of up to 45 m s^{-1} , although all measurements here are made at a nominal free-stream velocity of 20 m s^{-1} . Careful upstream flow conditioning and a three-dimensional contraction, with a reduction area ratio of 6.2, yields a

free-stream turbulence intensity ($\sqrt{u^2}/U_\infty$) of 0.05% at the start of the working section, and in the range of 0.15–0.2% at $x \approx 18$ m (for a free-stream velocity range of $20 < U_\infty < 40$ m s⁻¹). A series of adjustable slots and panels on the ceiling of the working section enables precise control of the streamwise pressure gradient. For all experiments presented here the coefficient of pressure C_p along the entire working section is constant to within $\pm 0.87\%$ (for all tripping configurations). Hence these experiments can be considered as concerning zero-pressure-gradient (ZPG) flows. Measurements are made in the turbulent boundary layer developing over the bottom wall of the working section. The 27 m long test surface permits the development of thick boundary layers. For example, with a free-stream velocity of $U_\infty \approx 20$ m s⁻¹, the boundary layer thickness varies from $\delta = 0.06$ m at $x = 1.6$ m from the tripped inlet, to $\delta = 0.32$ m at $x = 18$ m. The thick boundary layer enables us to attain good spatial resolution with a conventional hot-wire probe design.

Given the limited cross-sectional area of the working section (1.89×0.92 m²), compared to the very long length (27 m), careful attention was given to ensure that the boundary layers were nominally two-dimensional in the mean for the streamwise stations considered here. Nickels *et al.* (2005) report that, based on the studies of Jones, Marusic & Perry (1995) in a separate facility with adjustable side walls, the extra strain-rate effects due to finite working section width are negligible provided that the tunnel is wider than six boundary layer thicknesses (6δ), which is conservatively met to at least $x = 18$ m. Further tests are reported by Kulandaivelu (2012) who conducted a spanwise survey of free-stream velocity at $U_\infty \approx 20$ m s⁻¹ at $x = 10.5$ m over a spanwise distance of 0.8 m either side of the tunnel centreline, and found the variation to be less than $\pm 0.35\%$ with no distinguishable slope in the velocity variation across the width of the boundary layer (corresponding to over $\Delta y \approx 8\delta$ at this streamwise station). The wind tunnel also contains corner fillets throughout the facility. Comparisons of turbulence statistics at $x = 21$ m, with and without corner fillets in the working section, revealed no discernible differences, providing further confidence that the boundary layers at all streamwise locations reported here are nominally two-dimensional in the mean.

2.2. Hot-wire anemometry

All measurements were made using slightly modified Dantec 55P15 single-normal boundary layer type probes with prong tip spacing of 1.5 mm. Platinum–wollaston wires are soldered to the prong tips and etched to produce a 2.5 μ m diameter platinum sensing element of length $l \approx 0.54$ mm, resulting in a length to diameter ratio, $l/d \approx 215$ (see Ligrani & Bradshaw 1987; Hutchins *et al.* 2009). These sensors are driven in constant-temperature mode using in-house-designed Melbourne University Constant-Temperature Anemometers (MUCTA) with an overheat ratio of 1.8. The system response was verified using a square-wave electronic test to ensure a second-order response and demonstrating a -3 dB drop-off (based on the definition of Freymuth 1967) at 18 to 32 kHz for mean velocities in the range $0 < U < 20$ m s⁻¹. Hutchins *et al.* (2015) have recently shown that the temporal performance of all optimally tuned (under-damped) CTAs may start to diminish at frequencies lower than the -3 dB drop-off (at approximately 7–10 kHz). In this case, however, where high Reynolds numbers are attained with relatively low speeds and large development lengths, the frequency content of the turbulent boundary layer is relatively low. Hutchins *et al.* (2009) have shown that there is negligible energy content for $f^+ > 1/3$ (where $f^+ = f\nu/U_\tau^2$ and f is the frequency of turbulent

fluctuations in Hz), which corresponds to frequencies of approximately 10 kHz for these measurements. Hutchins *et al.* (2015) estimate the error due to frequency response in the peak variance of u at $zU_\tau/\nu = 15$ to be less than 0.5% for most anemometers provided that $U_\tau^2/\nu < 0.3 \times 10^{-5} \text{ s}^{-1}$. For the present experiments, $U_\tau^2/\nu \approx 0.35 \times 10^{-5} \text{ s}^{-1}$.

In terms of spatial resolution, since the unit Reynolds number is everywhere matched, the only variation in the viscous-scaled wire length l^+ occurs due to the weak variation in U_τ along the development length of the facility. At $x = 1.6$, the 0.54 mm long sensing element yields $l^+ = 25.6$, falling to 22.3 at $x = 18$ m. Hence, the probes can be considered to be nominally matched in terms of viscous length to within 24 ± 2 .

2.3. Data acquisition

The hot-wire signals were sampled at 65 kHz, using a Data Translations DT9836 analogue–digital converter. This equates to a viscous-scaled sample interval of $0.41 < \Delta t^+ < 0.53$ for the range of experiments ($\Delta t = 1/f_s$ is the time between samples, where f_s is sampling frequency). This exceeds the minimum timescale for energetic turbulent fluctuations (which is estimated at $t^+ \approx 3$, Hutchins *et al.* 2009), and thus the Nyquist criterion is met and aliasing is unlikely to be an issue. To be sure, the fluctuating signals are filtered at 25 kHz using a low-pass analogue filter. The total sampling time in seconds at each wall-normal position (z) is given by T . This is normalised in outer variables to give boundary layer turnover times TU_∞/δ . For converged statistics, these numbers need to be large, since the largest structures in turbulent boundary layers can exceed 20δ (e.g. Kim & Adrian 1999; Hutchins, Ganapathisubramani & Marusic 2004; Guala, Hommema & Adrian 2006; Hutchins & Marusic 2007*b*) and we would typically require several hundreds of these events to flow past the hot-wire sensor before we could expect converged statistics. In this study, the total sampling time was set in such a way that the boundary layer turnover time was in the range of 12 000–26 000 for all Reynolds numbers tested.

2.4. Calibration

The wires were calibrated statically against a Pitot-static tube located at the centreline of the tunnel in the undisturbed free stream. Calibrations are performed before and after each boundary layer traverse (referred to here as pre- and post-calibrations). To account for calibration drift during the experiments, the probe is periodically traversed to the free stream within the boundary layer profile measurement. At this free-stream excursion, the mean voltage measured by the hot wire and the mean velocity measured by the Pitot-static tube provides an additional re-calibration point at various intervals during the boundary layer traverse. Effectively, this means that for every six measurements during the boundary layer traverse (which consisted of between 33 and 50 logarithmically spaced measurement stations), the probe is re-calibrated. This procedure leads to a considerable reduction in the scatter between repeat experiments and is described in detail by Kulandaivelu (2012) and Talluru *et al.* (2014).

2.5. Tripping devices

Three flow cases were studied, corresponding to three different tripping configurations. All trips were introduced at the inlet to the working section (at $x = 0$ for the axis

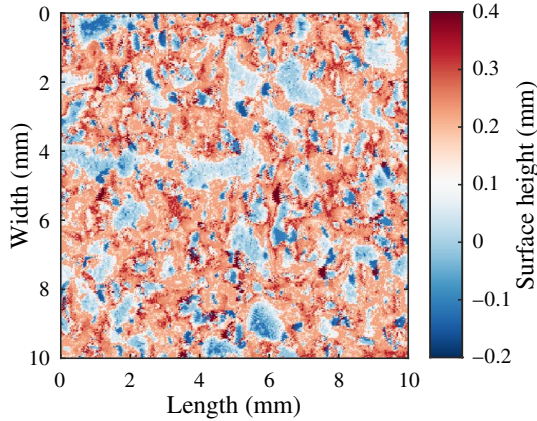


FIGURE 1. (Colour online) Contour plot depicting a scanned section (10 mm \times 10 mm) of the SP40 tripping configuration (P40 grit sandpaper). The colour bar on the right-hand side shows the distance from the mean plane (mm). Detailed specifications are given in table 1.

system used in this paper). The initial set of measurements was performed with the ‘standard’ tripping configuration, which consisted of a strip of P40 grit sandpaper, of length 154 mm in the streamwise direction. This trip was originally selected for this facility to obtain nominal shape factor values for ZPG turbulent boundary layers according to the recommendations of Erm & Joubert (1991). The next two tripping configurations were deliberately chosen to over-stimulate the boundary layer to study the effect of the trip on the evolution of mean flow parameters. In these configurations, 6 and 10 mm diameter threaded rods were added at $x=0$. We refer to the sandpaper and threaded rod configurations throughout as SP40, TR06 and TR10. Specifications of each trip configuration are given in table 1 and a representative scan of the sandpaper topography is shown in figure 1. To obtain consistent sets of measurements throughout the course of the investigation, reference conditions were fixed so that they corresponded to a given reference Reynolds number per metre. Here the nominal reference free-stream velocity for all three cases was 20 m s^{-1} , resulting in a reference Reynolds number per metre of $U_\infty/\nu = 1.295 \times 10^6 \text{ m}^{-1}$.

2.6. Streamwise pressure gradient

Since the pressure gradient plays an important role in the evolution of turbulent boundary layers, it is necessary to document the static pressure distribution along the length of the tunnel floor for all tripping configurations. The coefficient of pressure, C_p , along the working section is constant to within $\pm 0.87\%$ for SP40 configuration as indicated in figure 2. Necessary adjustments are made to the roof of the working section to achieve the desired zero pressure gradient. The static pressure distribution for TR10 and TR06 configurations are also in figure 2. Since the streamwise pressure distributions for all the tripping configurations exhibit negligible acceleration or deceleration of the flow, they can all be considered in a state of constant pressure.

3. Canonical reference case (SP40)

Table 2 collates the key experimental parameters for the set of matched-unit-Reynolds-number experiments conducted on the turbulent boundary layer developing

SP40: grit P40 sandpaper	
Commercial description	P40 grit sandpaper
Grit size	425–500 μm
Typical mean size (very coarse)	465 μm
Skewness	0.753 mm
Kurtosis	3.641 mm
Root-mean-square height	142 μm
Maximum peak height (height between the highest peak and the mean plane)	825 μm
Maximum pit height (depth between the mean plane and the deepest valley)	321 μm
Maximum height (height between the highest peak and the deepest valley)	1147 μm
Arithmetical mean height (mean surface roughness)	113 μm
Scanning details for SP40: configuration:	
Step size Y -direction	30 μm
Step size X -direction	30 μm
Percentage drop-outs ^a	1.67 %
TR10: 6 mm threaded rod	
Thread type	Male, single start
Major diameter	10.00 mm
Minor diameter	8.16 mm
Pitch	1.5 mm
TR06: 10 mm threaded rod	
Thread type	Male, single start
Major diameter	6.00 mm
Minor diameter	4.773 mm
Pitch	1 mm

TABLE 1. Specifications of the tripping configurations.

^aDrop-outs occur in clusters at or in between roughness elements on the sandpaper.

downstream of the standard P40 sandpaper trip. This represents what has previously been considered to be the canonical case ZPG turbulent boundary layer in the large Melbourne wind tunnel. The SP40 will be the reference measurement to compare with the over-stimulated cases in § 4. For all three configurations, the measured mean velocity U is fitted to the composite profile of Chauhan *et al.* (2009) to determine the local parameters U_τ , δ and Π . The log-law constants in the composite profile are $\kappa = 0.384$ and $B = 4.17$. Reynolds number and integral parameters listed in table 2, and table 3 in § 4, are calculated using the fitted composite profile. Figure 3 shows a schematic representation of the working section, showing the streamwise measurement stations, trip location and boundary layer development. Boundary layer traverses were conducted at 10 streamwise locations (S1 to S10) downstream of the trip, at $x = 1.60, 2.65, 3.75, 4.75, 6.3, 7.5, 10, 12.8, 17.5$ and 18.9 m. The local friction Reynolds number (Re_τ) together with local displacement-thickness Reynolds number ($Re_{\delta^*} = \delta^*U_\infty/\nu$) and the local momentum-thickness Reynolds number (Re_θ) for the SP40 trip case are also indicated for each measurement station.

Over these 10 measurement stations the boundary layer develops from a friction Reynolds number of $Re_\tau \approx 2700$ – $13\,000$. Such Reynolds numbers are significant

Station	x (m)	Re_τ	Re_{δ^*}	Re_θ	δ (mm)	δ_{99} (mm)	δ^* (mm)	θ (mm)	Π	U_∞^+	v/U_τ (μm)	l^+	Δr^+	TU_∞/δ
S1	1.60	2800	9800	7200	57.5	45.0	7.40	5.44	0.49	27.37	20.6	26.1	0.54	21000
S2	2.65	3600	12500	9300	75.2	60.1	9.42	6.99	0.48	27.97	21.1	25.5	0.52	16000
S3	3.75	4300	14900	11200	92.2	72.8	11.2	8.38	0.46	28.36	21.3	25.3	0.51	17500
S4	4.75	5100	16800	12700	108	81.9	12.6	9.49	0.41	28.54	21.3	25.3	0.51	16900
S5	6.30	6000	20600	15600	132	103	15.5	11.8	0.46	29.21	22.0	24.5	0.48	15300
S6	7.50	6700	22600	17200	150	115	17.2	13.1	0.43	29.36	22.4	24.1	0.46	16000
S7	10.00	8400	28400	21700	188	145	21.2	16.2	0.44	29.97	22.3	24.1	0.47	13100
S8	12.80	10500	33700	26100	237	176	25.2	19.5	0.38	30.25	22.6	23.9	0.45	15200
S9	17.50	13000	42800	33300	304	229	32.3	25.1	0.40	30.95	23.3	23.1	0.43	12100
S10	18.90	13400	45000	34900	319	242	34.4	26.7	0.43	31.15	23.8	22.7	0.41	11200

TABLE 2. Experimental parameters for the 10 profiles of the SP40 configuration. Bold notation for streamwise distance x indicates that it is a common measurement station with the TR06 and TR10 configurations in table 3.

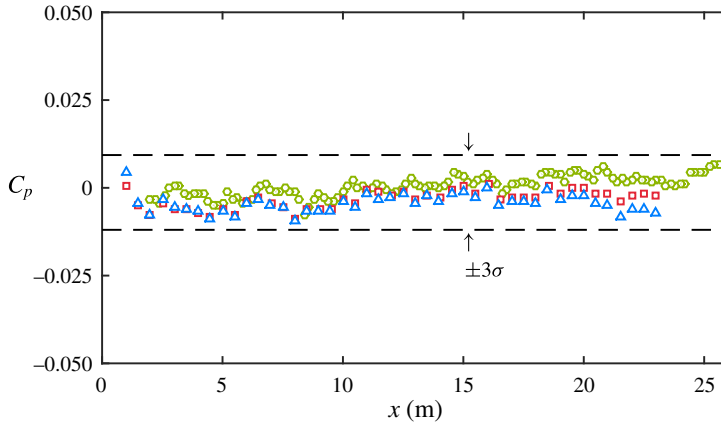


FIGURE 2. (Colour online) Streamwise pressure gradient for a reference free-stream velocity $U_\infty = 20 \text{ m s}^{-1}$. The downstream distance from the trip is given in the abscissa, whereas the ordinate denotes the variation of the coefficient of pressure in the x direction. \circ , SP40; \diamond , TR10; \square , TR06. The static pressure was measured at 20 cm and 50 cm intervals for the length of the tunnel floor for the SP40 configuration and TR10 & TR06 configurations, respectively.

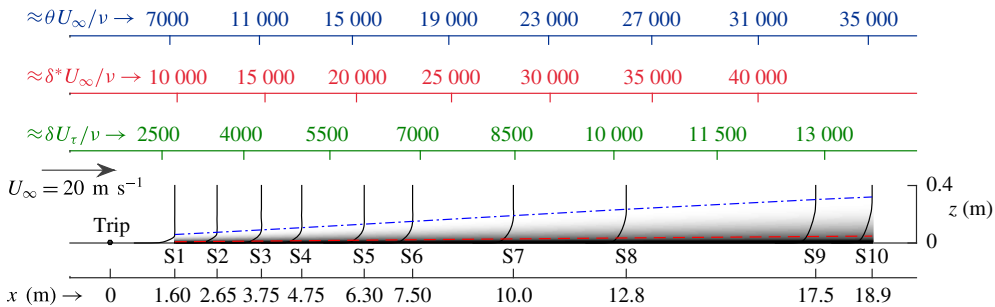


FIGURE 3. (Colour online) Schematic of the boundary layer development along the test section of the wind tunnel. S1 to S10 indicate measurement stations.

in terms of real-life engineering systems. For example, at the furthest downstream measurement station, the friction Reynolds number is equivalent to that obtained 6.7 m downstream of the nose (at the first passenger window) on a Gulfstream G550 cruising at an altitude of 14 500 m and at Mach 0.86 (Palumbo 2013). Figure 3 also shows the mean velocity profiles developing downstream of the P40 trip, plotted as U/U_∞ versus z , clearly demonstrating the growth in boundary layer thickness and fullness of the profile as x increases. The dash-dotted line on this plot shows the growth in boundary layer thickness δ , which exhibits a seemingly linear behaviour with x for this particular configuration. In this study the boundary layer thickness δ , the skin-friction velocity U_τ and the wake parameter Π are determined using the composite velocity profile fit of Chauhan *et al.* (2009).

Figure 3 also shows the contour of the outer limit of the overlap region, typically considered to be 0.15δ , as the dashed line. This is helpful in realising that in the actual measurements the overlap region spans a small fraction of the total boundary layer thickness. If the Reynolds number is increased by increasing the free-stream velocity

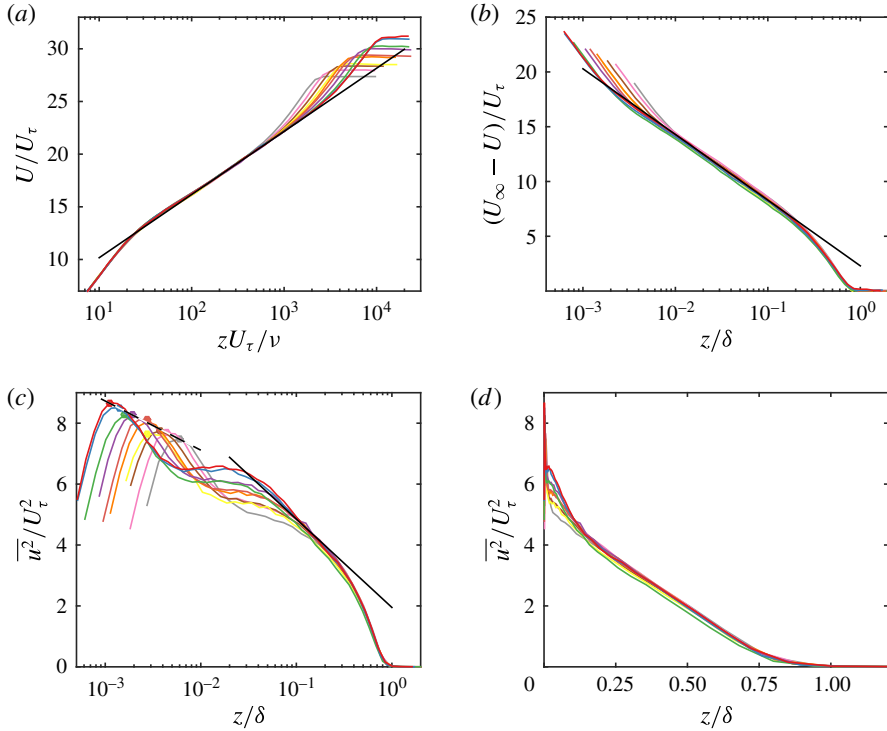


FIGURE 4. (Colour online) (a) Inner-normalised mean velocity profiles corresponding to the 10 streamwise hot-wire measurements acquired for the SP40 configuration. Straight line indicates the log-law $U^+ = 1/0.384 \ln(z^+) + 4.17$, (b) mean velocity defect profiles. Straight line corresponds to $(U_\infty - U)^+ = 2.3 - 1/0.384 \ln(z/\delta)$, (c) streamwise Reynolds normal stress $\overline{u^2}^+$ versus z/δ on log-scale. Filled symbols indicate the peak magnitude of $\overline{u^2}$ for each profile. The solid straight line corresponds to (3.1), and the dashed line is from (3.2), where the slope is as reported by Lee & Moser (2015), (d) $\overline{u^2}^+$ versus z/δ on linear axes.

then the outer limit of the log-law indicated in figure 3 will move further towards the wall, making the task of accurately resolving velocities in this region even more challenging.

Figure 4(a) shows the measured mean velocity profiles normalised in wall units. As mentioned above, the friction velocity U_τ is determined by fitting the data to the composite velocity profile of Chauhan *et al.* (2009) (with log-law constants $\kappa = 0.384$ and $B = 4.17$), and hence the collapse of the data on to the logarithmic law, marked by the straight line in figure 4(a), is forced. However, figure 4(a) demonstrates that the mean velocity profiles have a pronounced region of logarithmic behaviour at all measurement stations. This is reinforced by the velocity defect plots shown in figure 4(b) where a reasonable collapse to the logarithmic law is also seen for all profiles.

Figure 4(c) shows the profiles for $\overline{u^2}$ normalised by U_τ^2 against the outer-scaled wall-normal height z/δ that correspond to the 10 mean velocity profiles in figure 4(a) for the SP40 configuration. A good outer-scaling collapse is observed for nominally $z/\delta \geq 0.1$. For the logarithmic region the $\overline{u^2}/U_\tau^2$ profiles appear to reasonably well

follow the logarithmic behaviour

$$\frac{\overline{u^2}}{U_\tau^2} = B_1 - A_1 \ln(z/\delta), \quad (3.1)$$

as most recently discussed in Marusic *et al.* (2013), with $A_1 = 1.26$ and $B_1 = 1.95$. At higher Reynolds numbers the region over which (3.1) is valid increases (see figure 2 of Marusic *et al.* 2013). As expected, in the near-wall region, outer scaling does not apply; however, with inner scaling ($z^+ = zU_\tau/\nu$) a Reynolds-number independent scaling for $\overline{u^2}/U_\tau^2$ is not achieved either. The peak of $\overline{u^2}/U_\tau^2$ is seen to clearly increase with increasing Reynolds number, consistent with what has been extensively reported in the literature, e.g. DeGraff & Eaton (2000), Hutchins *et al.* (2009), Klewicki (2010), Marusic *et al.* (2010b) and others. The location at which the peak occurs (z_{max}) is considered to be fixed in the inner scaling, i.e. $z_{max}^+ \approx 15$ for ZPG turbulent boundary layers, and hence z_{max}/δ moves to the left with increasing Reynolds number in figure 4(c). The peaks are indicated by the filled symbols. A least-squares-error curve fit of these 10 points to the relation

$$\frac{\overline{u^2}_{max}}{U_\tau^2} = B_2 - A_2 \ln(z_{max}/\delta) \quad (3.2)$$

returns $A_2 = 0.7$ and $B_2 = 3.9$, which is generally consistent with results reported by Inoue *et al.* (2012). Recently, Lee & Moser (2015) surveyed DNS channel flow data up to $Re_\tau = 5200$, and reported a best fit of the peak variance results to be $\overline{u^2}_{max}^+ = 3.66 + 0.642 \ln(Re_\tau)$. The dashed line in figure 4(c) corresponds to (3.2) using the same slope as that of Lee & Moser (2015) ($A_2 = 0.642$; $B_2 = 4.26$), and it is seen to be in good agreement to within the experimental uncertainty of the measurements. It is noted that such a comparison of A_2 between DNS and the experiments here is only possible because we have maintained a fixed sensor length in wall units for all measurements ($l^+ \approx 24$). While the additive constant (B_2) is not expected to match the DNS results, as the limited sensor length will involve some attenuation of $\overline{u^2}^+$, the slope of (3.2) should not change with the reasonable assumption that the energy-contributing scales not resolved fully with a sensor of $l^+ \approx 24$ scale with viscous wall units (Chin *et al.* 2009; Marusic, Mathis & Hutchins 2010a). Similarly, figure 4(d) shows the profiles for $\overline{u^2}$ against the wall-normal distance z/δ in linear scale. The outer similarity of normalised $\overline{u^2}(z)$ with U_τ and δ as velocity and length scales, respectively, is evident in this figure.

The results presented so far for the SP40 configuration will suffice to establish that these measurements represent the canonical zero-pressure-gradient turbulent boundary layer and are in very good agreement with the known scaling behaviour for U and $\overline{u^2}$. Other results for this configuration will be discussed in comparison with the TR06 and TR10 configurations.

4. The over-stimulated boundary layers – TR06 and TR10

In this section we examine the evolution of the boundary layers that are ‘over-tripped’ by adding 6 and 10 mm threaded rods to the sandpaper trip (TR06 and TR10 configurations respectively). The mean flow parameters for TR06 and TR10 configurations are listed in table 3. Measurements were performed at eight and six streamwise locations for the 6 and 10 mm threaded rod trips, respectively.

SP40 station	x (m)	x/d_{trip}	Re_τ	Re_{δ^*}	Re_θ	δ (mm)	δ_{99} (mm)	δ^* (mm)	θ (mm)	Π	U_∞^+	v/U_τ (μm)	l^+	Δr^+	TU_∞/δ
TR06: 6 mm threaded rod															
S1	1.60	160	3 700	10 700	8 100	75.6	54.0	8.07	6.13	0.25	26.88	20.3	26.1	0.56	15 900
S2	2.65	265	4 900	13 500	10 400	101	72.0	10.1	7.78	0.21	27.40	20.5	25.8	0.55	18 200
S4	4.75	475	6 100	18 400	14 200	130	96.1	13.8	10.6	0.30	28.46	21.2	24.9	0.51	18 600
S5	6.30	630	6 800	22 100	16 900	148	113	16.5	12.6	0.39	29.20	21.8	24.3	0.48	16 400
S6	7.50	750	7 600	24 500	18 800	167	130	18.3	14.1	0.38	29.42	22.0	24.1	0.48	15 700
S8	12.80	1280	10 700	35 100	27 100	244	185	26.3	20.4	0.40	30.41	22.8	23.2	0.44	14 800
—	15.50	1550	12 300	40 000	31 100	284	213	20.0	23.3	0.38	30.71	23.0	23.0	0.43	14 900
—	22.10	2210	16 300	53 500	41 800	385	291	40.1	31.3	0.40	31.51	23.6	22.4	0.36	10 400
TR10: 10 mm threaded rod															
S1	1.60	160	8 000	13 900	11 000	161	84.8	10.4	8.27	—	26.75	20.1	26.8	0.57	7 500
S4	4.75	475	8 700	20 400	16 100	184	125	15.4	12.1	0.07	28.13	21.2	25.5	0.51	9 800
S6	7.50	750	9 300	26 200	20 500	203	150	19.7	15.4	0.24	29.19	21.9	24.6	0.48	11 900
S8	12.80	1280	11 800	36 700	28 600	269	205	27.6	21.5	0.34	30.36	22.8	23.6	0.44	13 400
—	15.50	1550	13 800	41 700	32 700	317	230	31.2	24.5	0.30	30.59	22.9	23.1	0.44	13 400
S10	18.90	1890	15 400	48 600	38 100	361	268	36.6	28.7	0.36	31.13	23.4	23.0	0.42	12 200

TABLE 3. Experimental parameters for eight and six profiles of the TR06 and TR10 configurations respectively. Bold notation for streamwise distance x indicates that it is a common measurement station for all three tripping configurations. d_{trip} corresponds to trip height.

The streamwise locations that are common to all three trip configurations (SP40, TR06 and TR10) are indicated by bold notation in the table. The boundary layer thickness δ , skin-friction velocity U_τ and the wake parameter Π are determined using the composite velocity profile fit of Chauhan *et al.* (2009) as was done for the SP40 configuration. The perturbations introduced by the threaded rods are more abrupt than those emerging from the sandpaper, resulting in larger boundary layer thicknesses at a given x . Since the incoming unit Reynolds number (U_∞/ν) is the same for all three configurations, Re_x is the same at a particular streamwise location. However the changes in δ (and integral thicknesses) result in the TR06 and TR10 configurations having a much higher friction Reynolds number than SP40 for $x \leq 10$ m. Hence these two configurations represent over-stimulated boundary layers.

4.1. Mean velocity

Figure 5(a) shows the mean velocity profiles using inner normalisation at four different streamwise locations. Profile measurements for all three trips are available at these four locations and hence a direct comparison can be made. The agreement of the U^+ versus z^+ profiles for all three configurations in the logarithmic region and below is expected since the inner normalisation utilises U_τ determined from a composite profile fit. However at Station S1 significant differences are observed in the outer wake region. Noticeably, U^+ in the wake region for SP40 is higher than TR06, which in turn is higher than TR10. Hence TR06 and TR10 configurations have a smaller value of Π than SP40 at Station S1. This systematic deviation persists further downstream at Stations S4 and S6, although the severity of the deviation is observed to lessen with increasing streamwise distance. At the last common station of Station S8 we observe that all three profiles are in good agreement with each other in terms of wake strength although the TR10 case has a greater boundary layer thickness than TR06 and SP40. Hence we can conclude that the threaded rods as transition triggers over-stimulated the boundary layer such that the scaling of the mean velocity in the wake region is altered. We emphasise here that the streamwise length required for the over-stimulated boundary layers to recover to a canonical state is specific to the flows considered here. A larger trip than TR10 will invariably require a longer recovery length.

In the inner region, all the profiles in figure 5(a) are seen to agree following the law of the wall. However, it should not be inferred that differences in the three cases (SP40, TR6 and TR10) are confined only to the outer region. The influence of a different trip is felt throughout the boundary layer and this is readily seen if one considers the defect profiles, as shown in figure 5(b). We find that at Station S1, the defect throughout the layer is highest for SP40 while it is relatively the lowest for the TR10 case. This implies that the profiles for TR06 and TR10 trips are ‘fuller’ and the mean velocities in the outer region are closer to the free stream in these cases. The systematic deviations observed in the defect profiles decrease with increasing x and eventually the outer similarity is regained by Station S8. It is only due to the change in corresponding U_τ that we see collapse in U^+ versus z^+ in the inner region. Outer scaling does not exist at low x because we have perturbed the boundary layer in the outer region.

Based on figure 5(a,b) we can conclude that an obvious effect of different trips is to modify the mean flow immediately downstream of the trip. The mean boundary layer in the three configurations becomes identical further downstream, as marked by

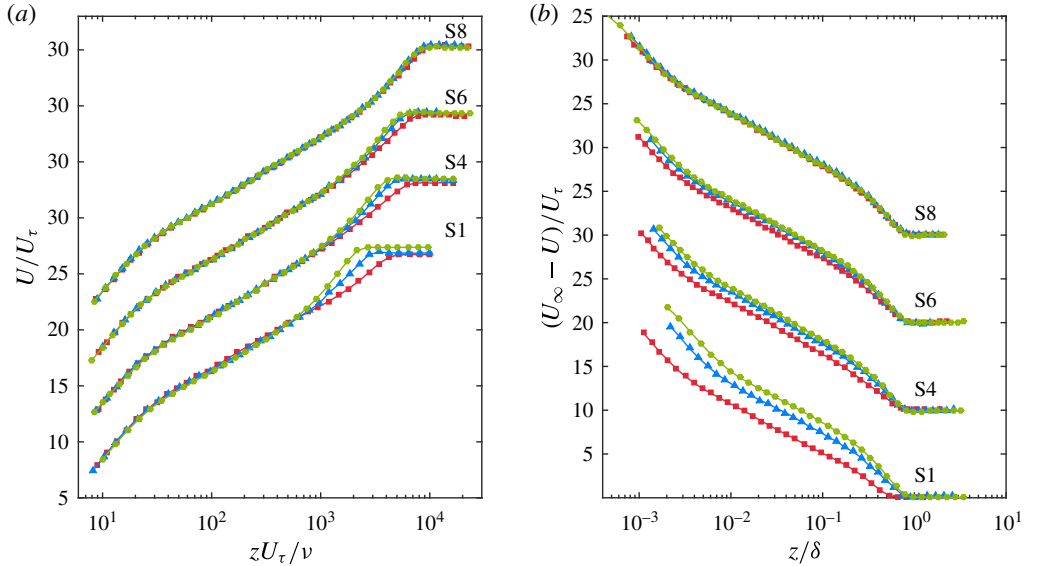


FIGURE 5. (Colour online) Comparison of mean velocity profiles for the three trips at four streamwise locations. \bullet , SP40; \blacktriangle , TR06; \blacksquare , TR10. Note the vertical shift in profiles.

the collapse of inner- and outer-scaled velocity profiles. The rate at which the over-stimulated boundary layers return to the canonical state depends on the incoming free-stream velocity and type of trip. In this study the TR06 configuration approaches the canonical state faster than the TR10. This point is elaborated on later.

4.2. Streamwise Reynolds normal stress

The changes to the streamwise Reynolds normal stress $\overline{u^2}$ are examined in figure 6. The widely accepted inner scaling by U_τ^2 for $\overline{u^2}$ is utilised here and again we find that the three profiles deviate considerably from each other at Station S1 downstream of the trip. The increased Re_τ values for the TR06 and TR10 cases are evident towards the edge of the layer where $\overline{u^2} \rightarrow 0$. In the logarithmic region, unlike U^+ versus z^+ in figure 5(a) where a collapse of profiles was observed, the $\overline{u^2}/U_\tau^2$ profiles are seen to diverge. Close to the wall (in the vicinity of $z^+ = 15$) the observed differences are relatively small, but the peak value for TR10 at Station S1 is distinctly higher. This indicates that the large-scale perturbations introduced in the outer part by the threaded rod change the structure of turbulence in the bulk of the layer, and the influence of these large scales appears to be felt well into the near-wall region during the initial development of the highly over-stimulated boundary layers (this will be shown more clearly when comparing spectra in § 4.4). Further downstream at Station S4 the deviation of profiles from each other is smaller than that seen at Station S1 but noticeably larger in the wake region. The profiles agree well with each other in the near-wall and logarithmic region at Station S6 but small differences still exist near the edge of the layer. It is clear that differences in the turbulence structure persist longer in the outer region, similarly to the differences in the mean velocity in figure 5(a). Finally, by Station S8 all profiles show an agreeable collapse and the three boundary layers appear to be similar. Thus we find that for the same Re_x (by keeping x the

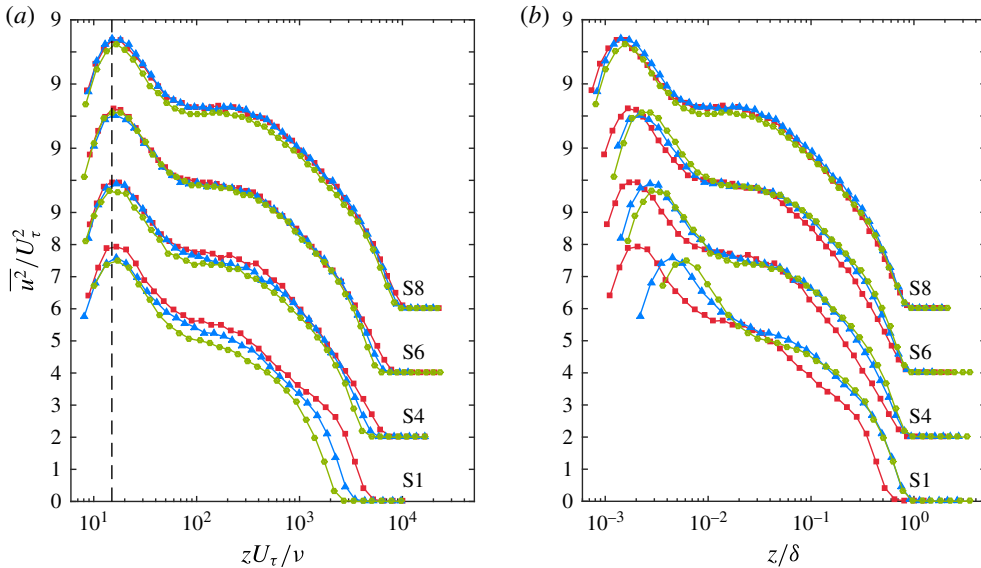


FIGURE 6. (Colour online) Comparison of $\overline{u^2}$ for the three trips at four streamwise locations. ●, SP40; ▲, TR06; ■, TR10. Note the vertical shift in profiles by 2 units.

same) the three trip configurations result in three different states of the boundary layer with respect to the structure of their mean velocity and streamwise normal Reynolds stress. Reynolds number similarity clearly does not hold in the downstream region of the trip. Only after a certain development length do the over-stimulated flows ‘relax’ to a state that can be classically considered as canonical. In our study, for the chosen trip configurations this development length is at least 12.8 m, corresponding to $Re_x = 1.7 \times 10^7$ and up to $O(2000)$ trip-height lengths.

4.3. Comparisons at matched local Reynolds number

Above, comparisons were shown at matched Re_x , and it is worthwhile to also consider the similarities and differences at matched local Reynolds numbers. Figure 7 shows comparisons of the above statistics at a nominally matched $Re_\tau \approx 8000$. For the mean flow, the profiles of SP40 at $x = 10$ m, and TR06 at $x = 7.5$ m are seen to match very well, although small differences remain for $\overline{u^2}^+$ suggesting that remnants of the initial tripping conditions persist. In the case of the highly over-tripped TR10 flow at $x = 4.75$ m both the mean velocity and the normal Reynolds stresses are seen to be notably different to the SP40 case, with a greatly reduced wake region for the mean flow, and a $\overline{u^2}^+$ profile that has a slightly smaller peak value at $z^+ \approx 15$ and a less full outer region, which at face value would be interpreted as a breakdown of Townsend’s outer-layer similarity hypothesis. Before proceeding further it is noted the comparisons in figure 7(a–c) are not for perfectly matched values of Re_τ . Therefore, the results of a check are shown in figure 7(d) where the streamwise stations of the three flows on either side of $Re_\tau = 8000$ were interpolated to obtain an estimated profile at $Re_\tau = 8000$. For example, the profile for TR10 in figure 7(d) was obtained by linear-interpolating the $\overline{u^2}^+$ profiles at matched values of z/δ for profiles at $x = 1.6$ m, $Re_\tau = 5850$ and $x = 4.75$ m, $Re_\tau = 8680$, weighted on Re_τ .

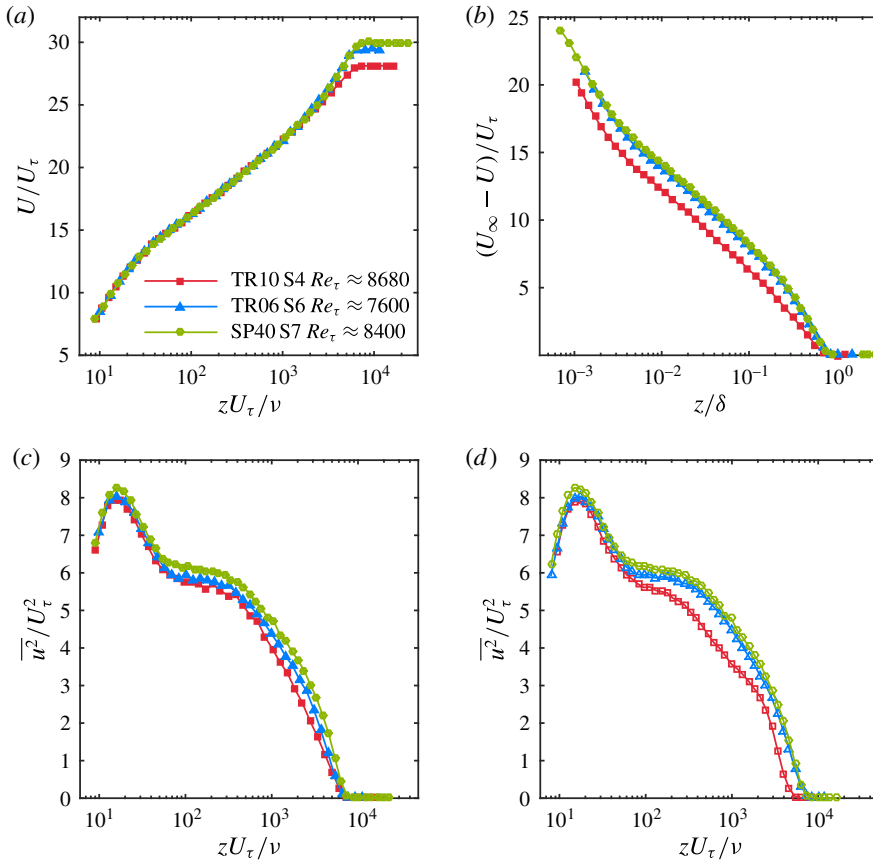


FIGURE 7. (Colour online) Comparison of statistics for the three flows (\bullet , SP40; \blacktriangle , TR06; \blacksquare , TR10) at a nominally matched $Re_\tau \approx 8000$. (d) A linear interpolation of data based on Re_τ between streamwise stations presented in tables 2 and 3 for $Re_\tau = 8000$.

The trends in figures 7(c) and 7(d) are seen to agree and therefore we conclude that the differences in the Re_τ values for the profiles in figure 7(a–c) is not the cause of the different trends noted. Rather, the lack of agreement between figures 7(c) and 7(d) is clearly due to the under-developed states of the over-stimulated boundary layers. This serves as a stark warning when comparing different experimental results based on matched Re_τ without consideration of the tripping conditions and evolution of the boundary layers. A further complication when considering Reynolds-number similarity in comparisons of different experimental results is what local Reynolds number to consider. In the wall turbulence literature Re_θ is commonly used and figure 8 shows the same comparisons as in figure 7 but here for nominally matched profiles at $Re_\theta \approx 10400$. Curiously, the results show that while the $\overline{u^2}^+$ profiles appear to be in nominal agreement, apart from a departure of the TR10 results in the outermost region of the flow, the mean flow profiles show distinct differences. These results further highlight the pitfalls of comparing statistics at matched Reynolds numbers if the boundary layers have not recovered from inappropriately chosen trips. In these cases it is clear that there is no one-to-one correspondence between Re_τ , Re_θ or Re_x . These issues will be considered further in §6.

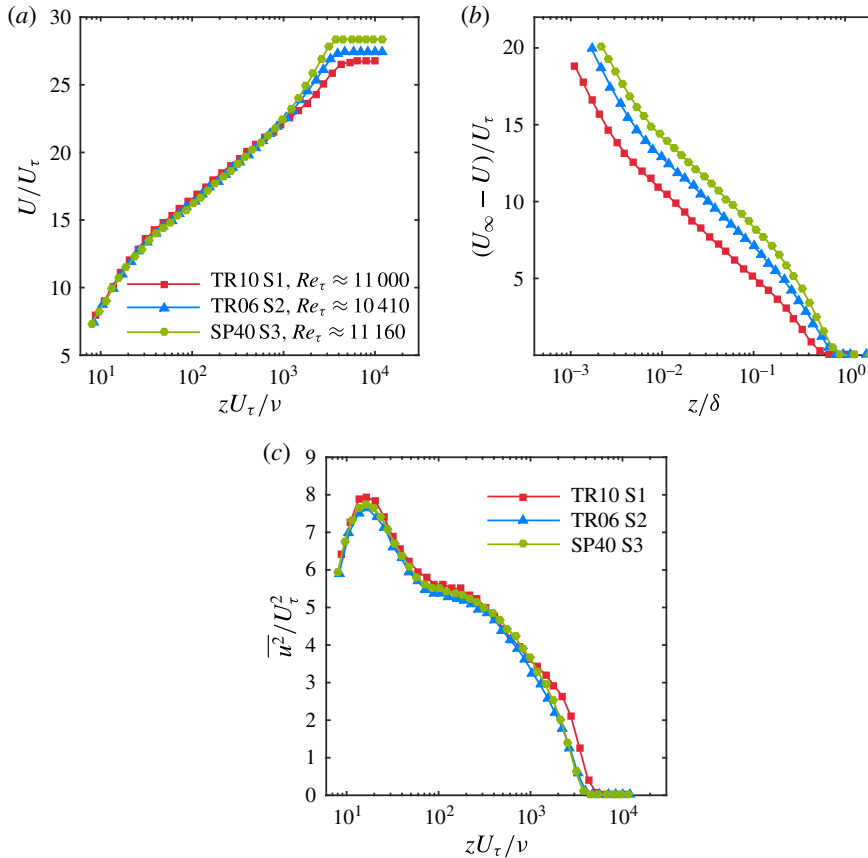


FIGURE 8. (Colour online) Comparison of statistics for the three flows (\bullet , SP40, $Re_\theta = 10\,680$; \blacktriangle , TR06, $Re_\theta = 10\,410$; \blacksquare , TR10, $Re_\theta = 10\,250$) at a nominally matched $Re_\theta \approx 10\,400$.

In order to gain insights into the differences in $\overline{u^2}$ observed above it is useful to consider the contributions from different length scales of turbulent motions using pre-multiplied spectra, and this is done in the following.

4.4. Pre-multiplied spectra

Figure 9 shows pre-multiplied spectra of streamwise velocity, $k_x\phi_{uu}$, as contour plots for each of the three trips at four streamwise locations, where the streamwise wavelength λ_x is used. At all x locations in figure 9 the similarity of $k_x\phi_{uu}/U_\tau^2$ is evident near the wall at all but the largest scales. For reference, ‘+’ is shown on each plot at $z^+ = 15$ and $\lambda_x^+ = 1000$, the reported location of the inner peak in the spectrogram (del Alamo *et al.* 2004; Hutchins & Marusic 2007a), which corresponds to the location of the inner peak of $\overline{u^2}/U_\tau^2$. The agreement seen in this near-wall region reinforces our interpretation that the influence of the wall is identical in these three boundary layers.

In contrast, in the outer region the pre-multiplied spectra are seen to have different characteristics, especially at smaller x locations. At Station S1, finite magnitudes of $k_x\phi_{uu}$ are present for TR06 and TR10 at higher z^+ than for SP40, consistent with

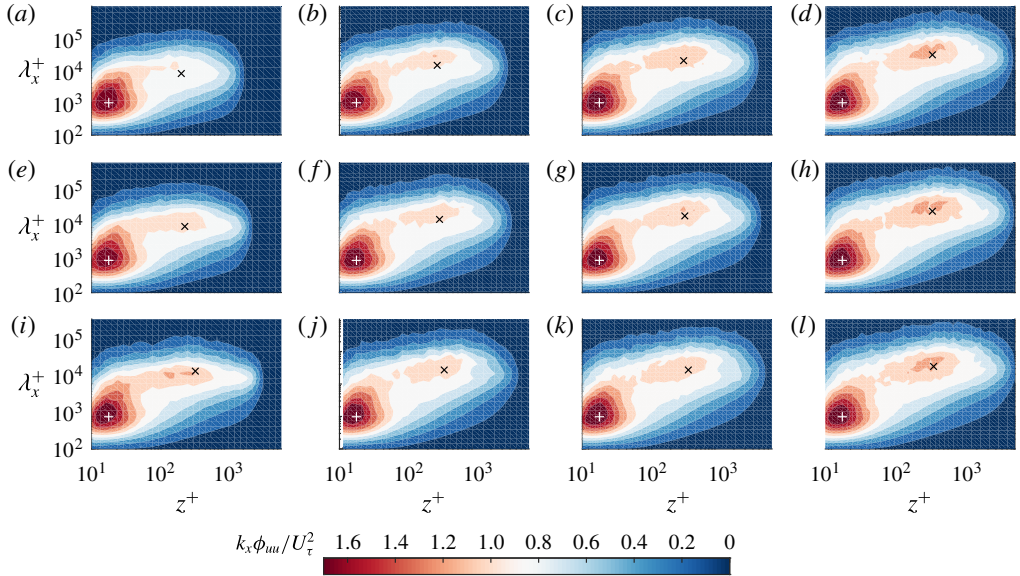


FIGURE 9. (Colour online) Pre-multiplied spectra of the streamwise velocity signal for the three trips. Each row is for a particular trip: (a–d) SP40, (e–h) TR06, (i–l) TR10. Each column is for a particular x location: (a,e,i) S1, (b,f,j) S4, (c,g,k) S6, (d,h,l) S8. + indicates $\lambda_x^+ = 1000$ and $z^+ = 15$, while \times indicates $\lambda_x/\delta = 6$ and $z^+ = 3.9Re_\tau^{1/2}$.

the increased Re_τ values. Hutchins & Marusic (2007a) reported that an outer peak in the u -spectrogram emerges at higher Reynolds numbers, and its location was later clarified by Mathis, Hutchins & Marusic (2009) to occur at $z^+ \approx 3.9Re_\tau^{1/2}$ and $\lambda_x/\delta \approx 6$; this location is marked by ‘ \times ’ for reference in all plots. The outer spectral peaks for the SP40 cases here appear to follow this quite well; however, it is noted that controversy remains as to the precise location and scaling of this outer spectral peak (see for example, Rosenberg *et al.* 2013; Vallikivi, Ganapathisubramani & Smits 2015) and the mark ‘ \times ’ is used here purely as a reference for three flows.

At Station S1 the outer peak is minimal for the SP40 configuration due to the low Reynolds number at this location. However, an outer peak does appear for the TR06 and TR10 configurations (more prominent for the TR10), but with the additional contributions being of shorter wavelength (closer to $\lambda_x^+ \approx 3Re_\tau$), and seemingly occurring close to the wall. These additional contributions explain the increased $\overline{u^2}/U_\tau^2$ observed in figure 6, and are consistent with the notion that the bigger trips have artificially aged and thickened the boundary layers. It is noted that the peak for TR10 appears distinctly while that for TR06 resembles a plateau that is connected to the inner peak. As the flows evolve downstream the outer peaks for the three trips seem to converge, and by Station S8 the spectrograms for all three trips appear similar in shape and magnitude.

It is deduced that the presence of a threaded rod at the trip location introduces large-scale disturbances in the low- Re boundary layer. These large-scale disturbances reside predominantly in the outer part of the boundary layer. A possible explanation is that these large scales either originate from or are amplified by the periodic shedding of the wake behind the rod. The presence of such energetic motions due to the abrupt tripping of the boundary layer is manifested as the outer peak in the spectrogram at low Re_x , while at the same Re_x such large scales are absent in the SP40 case.

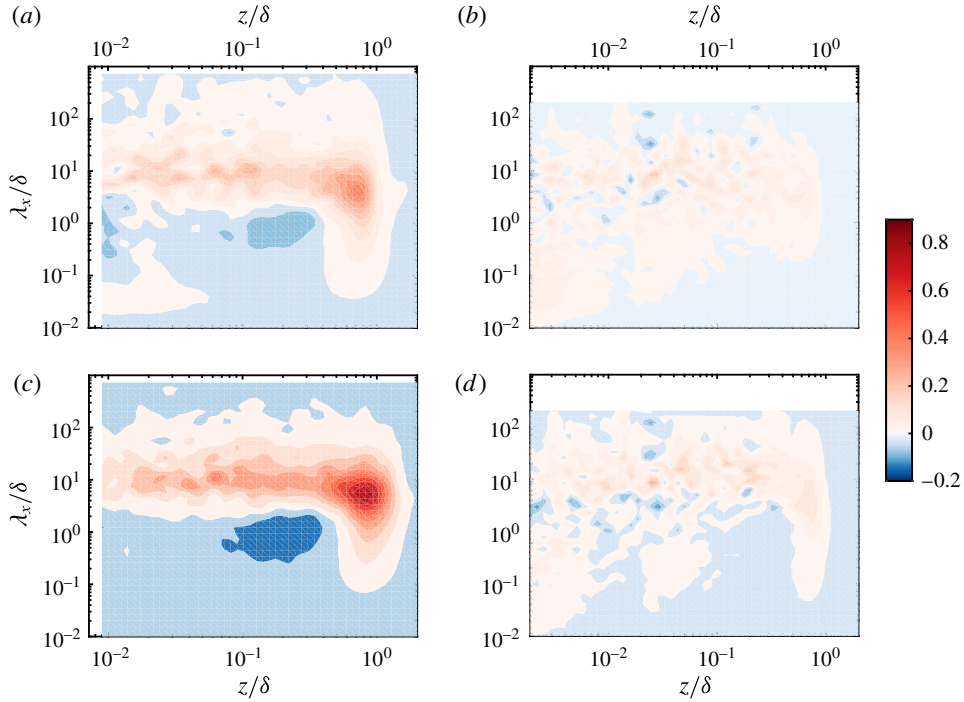


FIGURE 10. (Colour online) Excess energy content seen in pre-multiplied spectra of the streamwise velocity signal. Each column is for a particular x location: S1 (a,c) and S8 (b,d). Each row is for a particular difference between trip configurations: TR06 – SP40 (a,b) and TR10 – SP40 (c,d). Colour-bar levels indicate differences in $k_x \phi_{uu} / U_\tau^2$.

The differences observed between different trip configurations for the pre-multiplied spectra in figure 9 can be more clearly studied if we consider the difference in magnitude of $k_x \phi_{uu}$ at matched Re_x and at a particular z and λ_x . Such differences are plotted in figure 10. Figure 10(a) shows contours of the difference in $k_x \phi_{uu}$ between the SP40 and TR06 configurations at Station S1. The dimensional magnitude of $k_x \phi_{uu}$ is determined at a wall-normal height z and wavelength λ_x for the SP40 and TR06 spectrograms. Thereafter the magnitude of SP40 is subtracted from that of TR06. The result is non-dimensionalised using δ and U_τ of the SP40 configuration at Station S1. Similarly, figure 10(c) plots contours of the differences in spectrograms between SP40 and TR10 at Station S1, again utilising parameters of SP40 for non-dimensionalisation. It is clearly seen that the TR06 and TR10 configurations have significant excess kinetic energy in the outer region at Station S1. This excess energy is present in a wide range of scales $0.1 \leq \lambda_x / \delta \leq 100$ but pre-dominantly concentrated near $\lambda_x / \delta \approx 10$, with the peak difference occurring at $\lambda_x / \delta \approx 6$, $z / \delta \approx 0.8$, placing it well into the intermittent region of the boundary layer that typically spans $0.4 \leq z / \delta \leq 1$. Thus the threaded rod trips significantly influence the intermittent region of the boundary layer by introducing large-scale motions in this region. The excess of energy is larger for TR10 than for TR06, implying that the physical dimensions of the trip have a direct influence on the turbulence in the outer region. These observations for the scale-wise energy content in the outer part complement the observations for the changes in the mean velocity and streamwise normal stress seen earlier. The small

increase in the near-wall peak of $\overline{u^2}/U_\tau^2$ for TR10 at Station S1 is seen to come from the introduced large scales penetrating all the way down to the wall. This additional large-scale energy in the near-wall region is likely to also play a role in modulating the small scales near the wall (Mathis *et al.* 2009; Ganapathisubramani *et al.* 2012). An additional interesting feature to note in figure 10(a,c) is the region of reduced energy in the vicinity of $\lambda_x/\delta \approx 0.7$ and $z/\delta \approx 0.2$. This decrease in energy for the large scales near the outer edge of the log-region is not understood yet, but suggests a transfer of energy at the edge of the log-layer from wavelengths of order δ to much longer wavelengths.

Complementary to figure 10(a,c) we show the difference in scale-wise energy content at Station S8 in figure 10(b,d). As one would anticipate based on the agreement of mean velocity profiles in figure 5(a), variance in figure 6 and the spectrogram in figure 9(d,h,l), the differences in energy content at any particular wavelength are now quite small. However, for TR10, at Station S8, there is still a clearly observable, though now weakened, region of excess energy centred in the range $\lambda_x/\delta \approx 6\text{--}10$ and extending from the wall to the edge of the layer.

4.5. Higher-order moments

Above, we have observed that the effects of ‘over-tripping’ diminish as the boundary layers develop over the length of the plate, and as seen in figures 5 and 6, the second-order statistics appear to agree for different trips once both the mean velocity and defect velocities converge to the same respective profiles. This is perhaps surprising as one might intuitively expect that a higher-order statistic would require a longer development length to recover from a perturbed state (i.e. $\overline{u^2}$ should be more sensitive than U). As noted above, this does not appear to be the case. To test this further we consider higher-order statistics up to tenth order, but for brevity we only show the third- and eighth-order statistics to document the representative trends. These results are shown in figure 11 in the form of the skewness $S_u = \overline{u^3}/(\overline{u^2})^{3/2}$, and $(\overline{u^8})^{1/4}$ following the work of Meneveau & Marusic (2013), who showed that even moments represented in this way have a logarithmic behaviour with distance from the wall in the log-region of fully developed ZPG flows, and this is seen to be the case for the SP40 profiles. Comparison between the profiles in figures 11 and 6 indicates that the recovery length required for the statistics to become independent of the trip is not dependent on the order of the statistic (at least not up to tenth order), with all statistics nominally agreeing by Station S8. This suggests that while the larger trips introduce additional length scales into the flow, these perturbations and interactions relax as the boundary layer evolves downstream, and once they have decayed to the point of not further influencing the mean velocity profiles, their influence also appears to be negligible for the higher-order statistics. This finding implies that in order to determine if a flow has sufficiently recovered from a trip and reached a canonical ZPG boundary layer state, then only information about the evolution of the mean velocity profile is required. This is particularly advantageous as a reliable computation scheme can be developed for mean flow evolution, and this is considered in the following section.

5. Computing the evolution of turbulent boundary layers

The evolution of the mean flow for ZPG turbulent boundary layers at high Reynolds numbers can be computed in a number of ways, the most common approaches being RANS (Reynolds-averaged Navier–Stokes), CFD and large-eddy simulation. In those

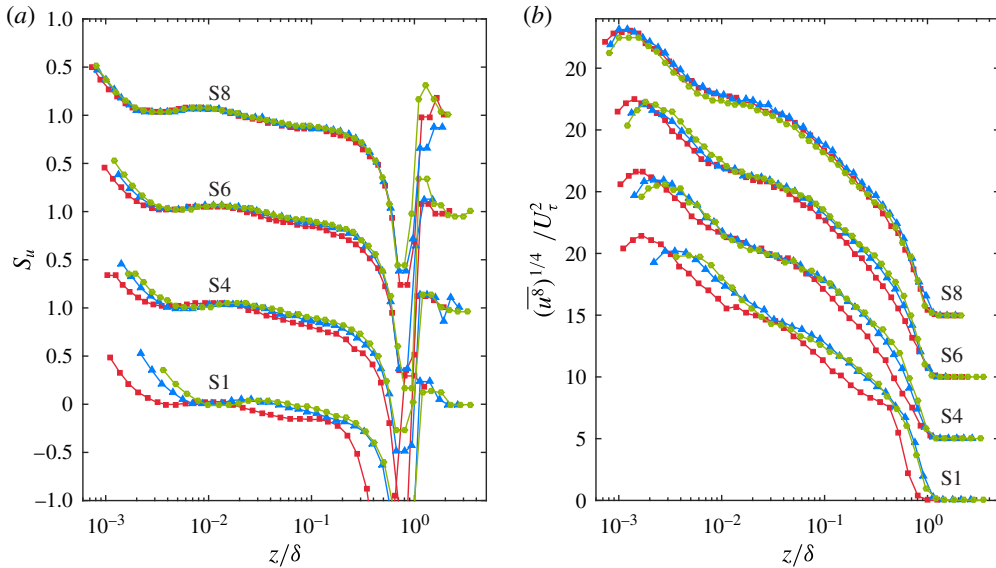


FIGURE 11. (Colour online) Comparison of skewness, $S_u = \overline{u^3} / (\overline{u^2})^{3/2}$, and $\overline{u^8}$ for the three trips at four streamwise locations. ●, SP40; ▲, TR06; ■, TR10. Note the vertical shift in profiles.

cases, however, models with multiple parameters are required together with a chosen grid, and the computations yield velocity profiles. Here, we are interested in mean flow parameters themselves (i.e. shape factor H , Coles wake factor Π etc.) and these can be directly computed using an integral equation approach described by Perry *et al.* (2002). For this one requires a closure formulation for the unknown Reynolds shear stresses to solve the Reynolds-averaged Navier–Stokes equations, and here we adopt a scheme similar to that developed by Perry *et al.* (1994). Jones, Marusic & Perry (2001) utilised such a closure formulation to evaluate the development of sink-flow boundary layers and showed good agreement of the calculated evolution with experiments. The objective here is to use the same set of closure formulations to compute the evolution of a boundary layer under zero pressure gradient from a given set of initial conditions.

The procedure involves the hypothesis that the total shear stress profile is uniquely described by a two-parameter family for a zero-pressure-gradient flow. In addition, a relation between the mean flow and shear stress parameters is required to close the system of equations. The equations that govern the streamwise evolution of a turbulent boundary layer can be found after considerable algebra by using the momentum integral and differential equations, the mean continuity equation, the log law of the wall and law of the wake. A more detailed explanation is given in Perry *et al.* (1994, 2002) and Jones *et al.* (2001) and therefore we only summarise the equations relevant to the results presented, for brevity.

5.1. Derivation of evolution equations

As described in Perry *et al.* (1994) and Perry & Marusic (1995), the total shear stress profile can be based on four non-dimensional mean flow parameters which describe

the state of the turbulent boundary layer:

$$\Pi, \quad S = \frac{U_\infty}{U_\tau}, \quad \beta = \frac{\delta^* dp}{\tau_0 dx}, \quad \zeta = S\delta \frac{d\Pi}{dx}. \quad (5.1a-d)$$

Here Π is the wake parameter, S is the skin friction parameter, β is the pressure gradient parameter, where p is the mean static pressure, and ζ is the non-equilibrium parameter. These parameters come from the mean momentum equation with the assumption that the mean velocity deficit profile is defined as a two-parameter family of the following form:

$$\frac{U_\infty - U}{U_\tau} = f\left(\frac{z}{\delta}, \Pi\right). \quad (5.2)$$

A coupled set of ordinary differential equations results (see Jones *et al.* 2001):

$$\frac{dS}{dRe_x} = \frac{\chi[Re_x, K] R[S, \Pi, \zeta, \beta]}{S E[\Pi] \exp[\kappa S]}, \quad (5.3)$$

$$\frac{d\Pi}{dRe_x} = \frac{\zeta \chi[Re_x, K]}{S^2 E[\Pi] \exp[\kappa S]}. \quad (5.4)$$

The functional forms of $R[S, \Pi, \zeta, \beta]$ and $E[\Pi]$ are given in detail in Jones *et al.* (2001) and Perry *et al.* (2002). Also,

$$\chi[R_x, K] = U_\infty(x)/U_0, \quad (5.5)$$

$$K = \frac{\nu}{U_\infty^2} \frac{dU_\infty}{dx} = 0, \quad (5.6)$$

where U_0 is the reference free-stream velocity (free-stream velocity at the streamwise location of the trip, i.e. $U_0 = U_\infty[x=0]$). In the case of ZPG flows $U_0 = U_\infty$, $\beta = 0$ and $K = 0$ since $dU_\infty/dx = 0$. Therefore, in summary, the evolution equations (5.3) and (5.4) are functions of three mean flow parameters only: S , Π and ζ . However, this leaves two equations with three unknowns. In order to close this problem, there needs to be one more equation, and in the absence of new theory that equation must be obtained empirically as a function of S , Π and ζ . This semi-empirical equation is termed the ‘closure equation’.

5.2. Closure formulation for ZPG flows

The restricted formulation of Perry *et al.* (1994) will now be extended with the effect of the parameter ζ included so that the non-equilibrium flow problem can be solved. Hence, following Perry *et al.* (2002),

$$F[\Pi, S, \beta, \zeta] = 0 \quad \Rightarrow \quad F[\Pi, S, \zeta] = 0. \quad (5.7)$$

It is assumed that no further parameters are involved in (5.7) and F is universal. Mapping out (5.7) from experimental data would help us to find the state of the developing boundary layer. As mentioned previously, the shear stress profile can be expressed by using these three mean flow parameters, Π , S and ζ . Figure 12 clearly shows that the variation of shear stress profiles with S is only marginal, even over a very large range of S . Since all the experimental data fall between S values of

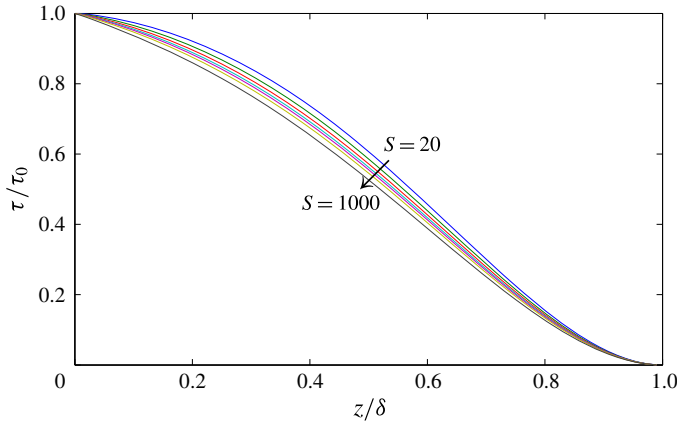


FIGURE 12. (Colour online) Total shear stress profiles computed using the closure formulation of Perry *et al.* (1994) for varying values of $S = U_\infty/U_\tau$.

20 to 40, it is very clear that we can approximate that the influence of S is small. Therefore, the variation in S is neglected and (5.7) is reduced to a two-parameter family of the following form:

$$F[\Pi, \zeta] \approx 0, \quad (5.8)$$

or

$$\zeta = f[\Pi], \quad (5.9)$$

which we will attempt to find empirically.

5.3. Log-law of the wall and wake formulation

Before determining the functional form of (5.9) a note on the definition of Π is necessary. One can find many log-law of the wall and wake formulations in the literature, e.g. Coles (1956), Moses (1964), Lewkowicz (1982), Jones (1998), Nickels (2004) and more recently Chauhan *et al.* (2009). As described earlier, the composite profile of Chauhan *et al.* (2009) was used to fit the experimental data in order to obtain the parameters Π , δ and U_τ , and therefore for consistency we continue to use their wall-wake formulation here, which is given by

$$\frac{U}{U_\tau} = U_{inner}^+(z^+) + \frac{2\Pi}{\kappa} W(z/\delta), \quad 0 \leq z \leq \delta, \quad (5.10)$$

where the inner function is essentially the near-wall formulation of Musker (1979) that approaches the logarithmic law with slope $\kappa = 0.384$ and additive constant $B = 4.17$ in the range $z^+ \geq 100$, and the wake function W takes the form of an exponential function. It should be noted, however, that any convenient formulation of the logarithmic profile coupled with a wake function can be used in the closure formulation of Perry *et al.* (1994) and Perry & Marusic (1995). The parameter Π in the theory is not bound to a particular choice of wake function.

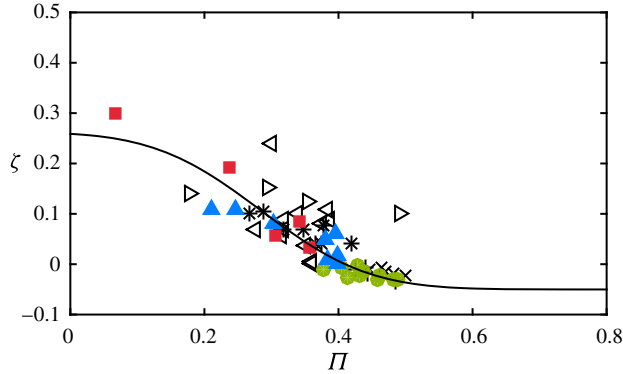


FIGURE 13. (Colour online) Variation of the non-equilibrium parameter ζ with wake parameter Π for experimental ZPG boundary layer measurements. Symbols are: *, Erm (1988); ▷, Klebanoff & Diehl (1951); ◁, Nagano *et al.* (1993); ×, $U_\infty = 37 \text{ m s}^{-1}$ data of Österlund (1999); +, $U_\infty = 42 \text{ m s}^{-1}$ data of Österlund (1999); ●, SP40; ▲, TR06; ■, TR10. Solid line is (5.12).

5.4. Functional form for ‘closure’ equation

In order to evaluate the evolution of the ZPG boundary layer a general expression is required for (5.9), i.e.

$$\zeta = f[\Pi]. \quad (5.11)$$

Although the attached eddy model of Perry *et al.* (1994) can assist us for quasi-equilibrium flows, at the present time we still require a model for finite ζ values. The answer can be found empirically provided enough experimental information is available. Here we require experiments where a constant free-stream velocity U_∞ was maintained at different streamwise stations x to obtain different Reynolds numbers, and unfortunately only a limited number of previous studies satisfy this requirement. In figure 13 we compile data that is available from the studies of Klebanoff & Diehl (1951), Erm (1988), Nagano, Tagawa & Tsuji (1993) and Österlund (1999), together with the present data from the three trips. A curve fit to (5.11) is obtained, given by

$$\zeta = 0.1862 \times \operatorname{erfc}[(\Pi - 0.2544) \times 4.935] - 0.0517. \quad (5.12)$$

Here the flow cases that have $\zeta > 0$ are identified as developing flows and the flow cases where $\zeta < 0$ are identified as relaxing flows. Previously Perry *et al.* (2002) proposed two closure equations corresponding to the relaxing and developing cases but it is deemed unnecessary for ZPG flow to have two functional forms for (5.11), as it is evident from figure 13 that all the discrete data appear to agree reasonably well with the proposed fit (5.12).

6. Comparison of computed evolution with experimental data

When comparing results from different sources it is important to have consistent definitions for the various measured quantities. Thus, in an effort to ease comparability, certain parameters such as δ , U_τ , H and C_f have been recomputed in a consistent manner for all data sets presented in this study. This is particularly important for the wake parameter Π . Figure 14 shows the evolution of mean flow parameters

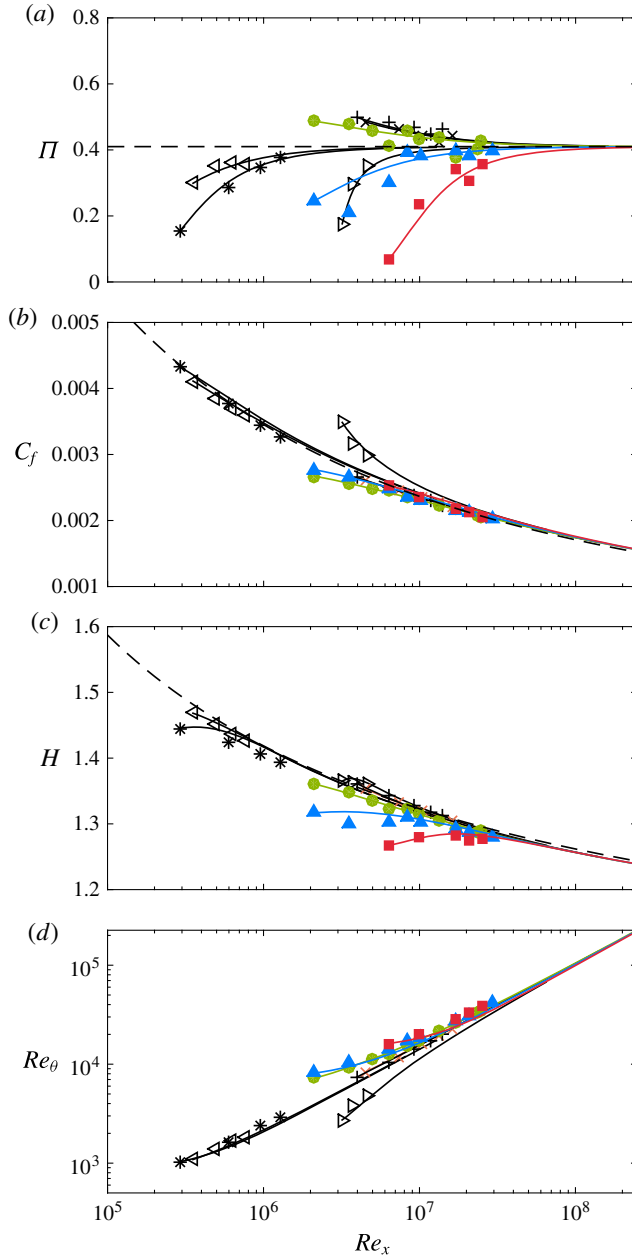


FIGURE 14. (Colour online) Comparison of experimental boundary layer parameters with computed evolution. (a) Π versus Re_x . Dashed line is the asymptotic limit of Π according to the definition of Chauhan *et al.* (2009); $\Pi \rightarrow 0.42$; (b) C_f versus Re_x . Dashed line is Schlichting's fit, $C_f = [2 \log_{10}(Re_x) - 0.65]^{-7/3}$; (c) H versus Re_x . Dashed line is $H = (1 - 7.1/\sqrt{2/C_f})^{-1}$; (d) Re_θ versus Re_x . See figure 13 for explanation of symbols.

compared with the predicted evolution solution computed using (5.3), (5.4) and (5.12). All measurements shown here were taken at a nominally constant free-stream velocity (U_∞) and the increase in Reynolds number is achieved by increasing x .

In order to compute the analytical evolution, the initial conditions for Π and S have to be specified, and this is done at the lowest-Reynolds-number measurement in a particular data set. For example, the initial conditions of the lowest-Reynolds-number measurement (at $Re_\tau = 2730$) are assigned to compute the analytical evolution for the tripping configuration SP40.

Figure 14(a) shows the evolution of Π and good agreement is observed between the experimental data and the computed evolutions (solid lines). For SP40, the flow is seen to be relaxing because the wake parameter starts at a slightly higher value initially and then approaches a constant as Re_x increases. This could be because the mean height of the roughness elements on the sandpaper trip are smaller than the incoming boundary layer height and therefore the transition is triggered within the layer, resulting in a weakly under-stimulated boundary layer (or the development of an internal layer). A cursory inspection of the mean velocity profiles does not seem to indicate this behaviour; however, the computed evolution captures this relaxing tendency very clearly. In the TR10 and TR06 cases, the flows are developing and as the Reynolds number increases the Π values increase and approach a constant. For both these trips the transition is triggered by introducing large-scale perturbations emerging from the wake behind the threaded rod. Thereby, the relative size of these perturbations is of the same order as the incoming boundary layer thickness. Hence these trips over-stimulate the flow and have fuller velocity profiles, resulting initially in smaller wake regions.

It is noted that the results in figure 14(a) show that even by Station S8 ($Re_x \approx 2 \times 10^7$) the Π values have not yet strictly converged. The computed evolution curves indicate that for the Π values for the three tripping configurations to all agree within 4% requires $Re_x \approx 7 \times 10^7$ (corresponding to $U_\infty^+ \approx 33$ and $x \approx 54$ m), which is well beyond the length of the facility. These results indicate that the trip does have a considerable effect in the wake region, not only for the low-Reynolds-number flows as suggested by Coles (1962) and Erm & Joubert (1991), but also at substantial Re_x . According to Coles (1962), Π is observed to drop for $Re_{\delta^*} < 5000$, and he described this as a low-Reynolds-number effect. However, here we see that this trend is better described as an evolution effect resulting from the initial conditions set up by the trip and/or the initial inflow conditions.

Also shown in figure 14(a) is the computed evolution for the data of Klebanoff & Diehl (1951), Erm (1988), Nagano *et al.* (1993) and Österlund (1999). In the case of Österlund's 32 m s^{-1} data and 43 m s^{-1} data, the flow is under-stimulated and the computed evolution predicts the Π trend reasonably well. The remaining data have over-stimulated flow and their corresponding development is again well predicted by the computation (solid lines). As a concluding comment for figure 14(a), it is found that any reasonable definition of Π used in the analysis of the experimental profiles does not change the trends seen in the figure, but only results in some shift in magnitude (value of Π) (see Chauhan, Nagib & Monkewitz 2007).

Figures 14(b), 14(c) and 14(d) show, respectively, the variation of friction coefficient $C_f = 2(U_\tau/U_\infty)^2$, shape factor $H = \delta^*/\theta$, and Reynolds number based on momentum thickness, Re_θ , versus Re_x , and again all the experiments considered in this study agree well with the computed evolution. The dashed line in figure 14(b) is Schlichting's relation modified according to Nagib *et al.* (2007); given as $C_f = [2 \log_{10}(Re_x) - 0.65]^{-7/3}$ (Schlichting 1960). Tripping configurations TR10 and TR06, where the flow is over-tripped, as well as SP40, where the boundary layer seems to be under-stimulated, exhibit behaviour that deviates from the canonical ZPG case at low Re_x . A distinctly noticeable deviation of C_f from the canonical behaviour

is also observed for the data of Klebanoff & Diehl (1951) (where a 0.04 in. rod was used as the trip). However, even in these cases the computed evolution still accurately predicts the C_f development.

The shape factor H is an integral parameter that can be considered as an independent criterion, since it offers a direct assessment of measured profiles without any assumptions or reliance on theory and independent wall-shear measurements. Figure 14(c) also shows the comparison between the equation for H provided in Chauhan *et al.* (2009) and the computed evolution for different experiments. The data points for Erm (1988) at low Reynolds number deviate from the dashed-curve of Chauhan *et al.* (2009) for canonical behaviour of H and likewise the non-canonical cases of TR06 and TR10 deviate considerably from the curve at high Reynolds numbers. However, the computed evolution captures the trend of the experimental data quite well. The comparison of Reynolds numbers in figure 14(d) emphasises that there is no one-to-one correspondence between Reynolds numbers as long as the remnants of the initial conditions persists in a given ZPG flow.

While the present experimental results are limited to one free-stream velocity, it is expected that similar qualitative trends will emerge for other free-stream velocities and that an account of the boundary layer evolution is needed in all cases. The overall results in figure 14 highlight that matching one Reynolds number does not guarantee similar states of two different ZPG flows. It is clear that non-equilibrium effects introduced in the boundary layer at any stage not only cause the flow to deviate from the canonical state but also have an influence on its growth immediately downstream at low Reynolds number. The good news, though, is that the above results indicate that the streamwise evolution of ZPG turbulent boundary layers can be computed, provided the appropriate upstream initial conditions are specified.

7. Conclusions

Measurements were performed in the large University of Melbourne wind tunnel to study the spatial evolution of zero-pressure-gradient turbulent boundary layers from their origin to a high-Reynolds-number state. Careful attention was given to the experimental procedures in order to make comparisons between flows with three different trips while keeping all other parameters nominally constant, including the pressure gradient and keeping the measurement sensor size nominally fixed at $l^+ \approx 24$. The three trips consist of a standard sandpaper trip (SP40) and two cases where additional threaded rods of 6 mm (TR06) and 10 mm (TR10) diameter are added. The SP40 case was regarded as the standard case and measurements at 10 different streamwise stations covering a range $2700 < Re_\tau < 13\,000$ showed good agreement with the known scaling behaviour for mean flow and streamwise Reynolds stresses described in the literature (Smits, McKeon & Marusic 2011). The near-wall peak value in $\overline{u^{2+}}$ at $z^+ \approx 15$ is found to follow a log-linear dependence with Re_τ with a slope consistent with that recently reported from DNS results by Lee & Moser (2015).

Since U_∞/ν was held constant for all cases, comparisons at matched streamwise distance from the trip (x) correspond to matched Re_x . The TR06 and TR10 cases were found to have significant differences to the SP40 case for U^+ and $\overline{u^{2+}}$ at low Reynolds number, particularly in the outer region of the boundary layer, with the larger trip (TR10) resulting in larger deviations from canonical behaviour. Comparisons of spectra reveal that the over-stimulated trips introduce large-scale disturbances into the boundary layer, which are prominent at low Re . These large-scale disturbances reside

predominantly in the outer part of the boundary layer, and probably originate from or are amplified by the periodic shedding of the wake behind the rod. The presence of such energetic motions due to the abrupt tripping of the boundary layer is manifested as an outer peak in the spectrogram at low Re_x , while at the same Re_x such large scales are absent in the SP40 case. This artificial outer peak is different to the naturally occurring outer spectral peak that occurs at high Re in canonical ZPG flows. The remnants of the ‘over-tripped’ conditions are seen to persist for some distance, at least until Station S8, at which position the non-canonical boundary layers (TR06 and TR10) exhibit a weak memory of their initial conditions only for the large scales $O(10\delta)$. Generally, the mean velocity profiles were found to approach a constant wake parameter Π as the three boundary layers developed along the test section, and the mean flow, broadband turbulence intensities, and higher-order moments are all found to nominally agree by Station S8.

A comparison of U^+ and $\overline{u^+}$ was also made at nominally matched local Reynolds numbers, namely at $Re_\tau \approx 8000$ and $Re_\theta \approx 10400$. In both cases distinct differences were observed despite the matched local Reynolds number, due to the incomplete recoveries of the over-stimulated boundary layers. In these cases it is clear that there is no one-to-one correspondence between Re_τ , Re_θ or Re_x , and this highlights that caution is needed when comparing data sets obtained under different free-stream velocity conditions to achieve different Reynolds numbers, with little or no information about their initial or upstream boundary conditions. Unfortunately the majority of studies in the literature fall into this category. We find that non-equilibrium effects in the boundary layer are not confined to low Reynolds number, and can persist even at high Reynolds numbers. All ZPG boundary layers would be expected to evolve to an equivalent form where Reynolds-number similarity holds, provided the Reynolds number is sufficiently high, and the larger the trip disturbance is above its ideal trip size the longer the boundary layer will need to recover. However, it is not clear *a priori* how long this will take and how far downstream of the trip this will occur. For the trips considered in our study, the recovery occurs at $Re_x \approx 1.7 \times 10^7$, which corresponds to a distance along the plate of $O(2000)$ trip heights. It is also noted that the present study is restricted to measurements only of the streamwise component of velocity. Seo *et al.* (2004) have shown that wall-normal component statistics are more sensitive to initial conditions, such as $\overline{w^2}$ compared to $\overline{u^2}$, and this may change the quantitative values of the evolution lengths required to recover from over-tripped conditions, and/or the conclusions regarding statistics of other velocity components.

A perhaps surprising but important finding from the present study is that the recovery length required for the statistics to become independent of the trip is not dependent on the order of the statistic (at least not up to tenth order for u). This suggests that, while the larger trips introduce additional length scales into the flow, these perturbations and interactions relax as the boundary layer evolves downstream and once they have decayed to the point of not further influencing the mean velocity profiles, their influence also appears to be negligible for the higher-order statistics. This is at least the case in the present study where the inflow initial conditions are maintained constant and only the trip is varied. This result therefore implies that for streamwise velocity statistics it may be sufficient to document the mean flow parameters alone in order to ascertain whether the ZPG flow has reached a canonical state.

The evolution of the mean flow parameters is considered by computing solutions of the momentum integral equations using a semi-empirical closure relation for ZPG

turbulent boundary layers based on a survey of available data. Using this approach, the predicted evolution for all tripped configurations is found to be in agreement with the experimental data for the mean flow parameters such as Π , H , C_f for a wide range of Reynolds numbers. From figure 14, it can be seen that, though the local parameters such as the Reynolds number Re_x are matched (for the sake of Reynolds-number similarity), different mean flow parameters (Π , H , C_f) can be obtained. This computational scheme should therefore be useful as a tool towards determining whether a given ZPG boundary layer is of a canonical form.

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REFERENCES

- DEL ALAMO, J. C., JIMÉNEZ, J., ZANDONADE, P. & MOSER, R. D. 2004 Scaling of the energy spectra of turbulent channels. *J. Fluid Mech.* **500**, 135–144.
- CASTILLO, L. & JOHANSSON, T. G. 2002 The effects of the upstream conditions on a low Reynolds number turbulent boundary layer with zero pressure gradient. *J. Turbul.* **3** (31), 1–19.
- CASTILLO, L. & WALKER, D. 2002 Effect of upstream conditions on the outer flow of turbulent boundary layers. *AIAA J.* **40** (7), 1292–1299.
- CHAUHAN, K. A. & NAGIB, H. M. 2008 On the development of wall-bounded turbulent flows. In *IUTAM Symposium on Computational Physics and New Perspectives in Turbulence*, pp. 183–189. Springer.
- CHAUHAN, K. A., NAGIB, H. M. & MONKEWITZ, P. A. 2007 On the composite logarithmic profile in zero pressure gradient turbulent boundary layers. In *45th AIAA Aerospace Sciences Meeting and Exhibit, Reno, NV*, vol. 1, pp. 532–549.
- CHAUHAN, K. A., NAGIB, H. M. & MONKEWITZ, P. A. 2009 Criteria for assessing experiments in zero pressure gradient boundary layers. *Fluid Dyn. Res.* **41**, 021404.
- CHIN, C. C., HUTCHINS, N., OOI, A. S. H. & MARUSIC, I. 2009 Use of direct numerical simulation (DNS) data to investigate spatial resolution issues in measurements of wall-bounded turbulence. *Meas. Sci. Technol.* **20**, 115401.
- COLES, D. E. 1956 The law of the wake in the turbulent boundary layer. *J. Fluid Mech.* **1**, 191–226.
- COLES, D. E. 1962 The turbulent boundary layer in a compressible fluid. Appendix A: A manual of experimental boundary-layer practice for low-speed flow. *Tech. Rep.* R-403-PR. USAF The Rand Corporation.
- DEGRAFF, D. B. & EATON, J. K. 2000 Reynolds-number scaling of the flat-plate turbulent boundary layer. *J. Fluid Mech.* **422**, 319–346.
- ERM, L. P. 1988 Low-Reynolds-number turbulent boundary layers. PhD thesis, The University of Melbourne, Melbourne, Australia.
- ERM, L. P. & JOUBERT, P. N. 1991 Low-Reynolds-number turbulent boundary layer. *J. Fluid Mech.* **230**, 1–44.
- FREYMUTH, P. 1967 Feedback control theory for constant-temperature hot-wire anemometers. *Rev. Sci. Instrum.* **38** (5), 677–681.
- GANAPATHISUBRAMANI, B., HUTCHINS, N., MONTY, J. P., CHUNG, D. & MARUSIC, I. 2012 Amplitude and frequency modulation in wall turbulence. *J. Fluid Mech.* **712**, 61–91.
- GEORGE, W. K. & CASTILLO, L. 1997 Zero pressure gradient turbulent boundary layer. *Appl. Mech. Rev.* **50**, 689–729.
- GUALA, M., HOMMEMA, S. E. & ADRIAN, R. J. 2006 Large-scale and very-large-scale motions in turbulent pipe flow. *J. Fluid Mech.* **554**, 521–542.
- HUTCHINS, N. 2012 Caution: tripping hazards. *J. Fluid Mech.* **710**, 1–4.

- HUTCHINS, N., GANAPATHISUBRAMANI, B. & MARUSIC, I. 2004 Dominant spanwise Fourier modes, and the existence of very large scale coherence in turbulent boundary layers. In *15th Australasian Fluid Mechanics Conference, Sydney, Australia*, AFMS.
- HUTCHINS, N. & MARUSIC, I. 2007a Evidence of very long meandering features in the logarithmic region of turbulent boundary layers. *J. Fluid Mech.* **579**, 1–28.
- HUTCHINS, N. & MARUSIC, I. 2007b Large-scale influences in near-wall turbulence. *Phil. Trans. R. Soc. Lond. A* **365**, 647–664.
- HUTCHINS, N., MONTY, J. P., HULTMARK, M. & SMITS, A. J. 2015 A direct measure of the frequency response of hot-wire anemometers: temporal resolution issues in wall-bounded turbulence. *Exp. Fluids* **56** (1), 1–18.
- HUTCHINS, N., NICKELS, T. B., MARUSIC, I. & CHONG, M. S. 2009 Hot-wire spatial resolution issues in wall-bounded turbulence. *J. Fluid Mech.* **635**, 101–136.
- INOUE, M., MATHIS, R., MARUSIC, I. & PULLIN, D. I. 2012 Inner-layer intensities for the flat-plate turbulent boundary layer combining a predictive wall-model with large-eddy simulations. *Phys. Fluids* **24** (7), 075102.
- JOHANSSON, G. & CASTILLO, L. 2001 LDA measurements in turbulent boundary layers with zero pressure gradient. In *Proc. 2nd Int. Symp. Turbulent Shear Flow Phenomena, Stockholm, Sweden*.
- JONES, M. B. 1998 Evolution and structure of sink flow turbulent boundary layers. PhD thesis, The University of Melbourne, Melbourne, Australia.
- JONES, M. B., MARUSIC, I. & PERRY, A. E. 1995 The effect of aspect ratio and divergence on the turbulence structure of boundary layers. In *Proceedings of the 12th Australasian Fluid Mech. Conf., Sydney, Australia*, pp. 436–439.
- JONES, M. B., MARUSIC, I. & PERRY, A. E. 2001 Evolution and structure of sink-flow turbulent boundary layers. *J. Fluid Mech.* **428**, 1–27.
- KIM, K. C. & ADRIAN, R. J. 1999 Very large-scale motion in the outer layer. *Phys. Fluids* **11**, 417–422.
- KLEBANOFF, P. S. & DIEHL, Z. W. 1951 Some features of artificially thickened fully developed turbulent boundary layers with zero pressure gradient. *Tech. Rep.* 2475. DTIC Document.
- KLEWICKI, J. C. 2010 Reynolds number dependence, scaling, and dynamics of turbulent boundary layers. *J. Fluids Engng* **132** (9), 094001.
- KULANDAIVELU, V. 2012 Evolution of zero pressure gradient turbulent boundary layers from different initial conditions. PhD thesis, University of Melbourne.
- LEE, M. K. & MOSER, R. D. 2015 Direct numerical simulation of a turbulent channel flow up to $Re_\tau = 5200$. *J. Fluid Mech.* **744**, 395–415.
- LEWKOWICZ, A. K. 1982 An improved universal wake function for turbulent boundary layers and some of its consequences. *Z. Flugwiss. Weltraumforsch.* **6**, 261–266.
- LIGRANI, P. M. & BRADSHAW, P. 1987 Spatial resolution and measurement of turbulence in the viscous sublayer using subminiature hot-wire probes. *Exp. Fluids* **5**, 407–417.
- MARUSIC, I., MATHIS, R. & HUTCHINS, N. 2010a High Reynolds number effects in wall turbulence. *Intl J. Heat Fluid Flow* **31**, 418–428.
- MARUSIC, I., MCKEON, B. J., MONKEWITZ, P. A., NAGIB, H. M., SMITS, A. J. & GREENIVASAN, K. R. 2010b Wall-bounded turbulent flows at high Reynolds numbers: Recent advances and key issues. *Phys. Fluids* **22** (6), 065103.
- MARUSIC, I., MONTY, J. P., HULTMARK, M. & SMITS, A. J. 2013 On the logarithmic region in wall turbulence. *J. Fluid Mech.* **716**, R3.
- MATHIS, R., HUTCHINS, N. & MARUSIC, I. 2009 Large-scale amplitude modulation of the small-scale structures in turbulent boundary layers. *J. Fluid Mech.* **628**, 311–337.
- MENEVEAU, C. & MARUSIC, I. 2013 Generalized logarithmic law for high-order moments in turbulent boundary layers. *J. Fluid Mech.* **719**, R1.
- MONKEWITZ, P. A., CHAUHAN, K. A. & NAGIB, H. M. 2007 Self-contained high Reynolds-number asymptotics for zero-pressure-gradient turbulent boundary layers. *Phys. Fluids* **19**, 115101.
- MONKEWITZ, P. A., CHAUHAN, K. A. & NAGIB, H. M. 2008 Comparison of mean flow similarity laws in zero pressure gradient turbulent boundary layers. *Phys. Fluids* **20**, 105102.

- MOSES, H. L. 1964 The behavior of turbulent boundary layers in adverse pressure gradients. *Tech. Rep. 73*. Gas Turbine Lab. MIT.
- MUSKER, A. J. 1979 Explicit expression for the smooth wall velocity distribution in a turbulent boundary layer. *AIAA J.* **17** (6), 655–657.
- NAGANO, Y., TAGAWA, M. & TSUJI, T. 1993 Effects of adverse pressure gradients on mean flows and turbulence statistics in a boundary layer. In *Turbulent Shear Flows 8*, pp. 7–21. Springer.
- NAGIB, H. M., CHAUHAN, K. A. & MONKEWITZ, P. A. 2007 Approach to an asymptotic state for zero pressure gradient turbulent boundary layers. *Phil. Trans. R. Soc. Lond. A* **365**, 755–770.
- NICKELS, T. B. 2004 Inner scaling for wall-bounded flows subject to large pressure gradients. *J. Fluid Mech.* **521**, 217–239.
- NICKELS, T. B., MARUSIC, I., HAFEZ, S. & CHONG, M. S. 2005 Evidence of the k_1^{-1} law in a high-Reynolds-number turbulent boundary layer. *Phys. Rev. Lett.* **95**, 074501.
- ÖSTERLUND, J. M. 1999 Experimental studies of zero-pressure gradient turbulent boundary layer flow. PhD thesis, Royal Institute of Technology, Stockholm, Sweden.
- PALUMBO, D. 2013 The variance of convection velocity in the turbulent boundary layer and its effect on coherence length. *J. Sound Vib.* **332** (15), 3692–3705.
- PERRY, A. E. & MARUSIC, I. 1995 A wall-wake model for the turbulence structure of boundary layers. Part 1. Extension of the attached eddy hypothesis. *J. Fluid Mech.* **198**, 361–388.
- PERRY, A. E., MARUSIC, I. & JONES, M. B. 2002 On the streamwise evolution of turbulent boundary layers in arbitrary pressure gradients. *J. Fluid Mech.* **461**, 61–91.
- PERRY, A. E., MARUSIC, I. & LI, J. D. 1994 Wall turbulence closure based on classical similarity laws and the attached eddy hypothesis. *Phys. Fluids* **6** (2), 1024–1035.
- ROSENBERG, B. J., HULTMARK, M., VALLIKIVI, M., BAILEY, S. C. C. & SMITS, A. J. 2013 Turbulence spectra in smooth- and rough-wall pipe flow at extreme Reynolds numbers. *J. Fluid Mech.* **731**, 46–63.
- SCHLATTER, P. & ÖRLÜ, R. 2010 Assessment of direct numerical simulation data of turbulent boundary layers. *J. Fluid Mech.* **659**, 116–126.
- SCHLATTER, P. & ÖRLÜ, R. 2012 Turbulent boundary layers at moderate Reynolds numbers: inflow length and tripping effects. *J. Fluid Mech.* **710**, 5–34.
- SCHLICHTING, H. 1960 *Boundary-layer Theory*. McGraw-Hill.
- SEO, J., CASTILLO, L., JOHANSSON, T. G. & HANGAN, H. 2004 Reynolds stress in turbulent boundary layers at high Reynolds number. *J. Turbul.* **5**, 1–13.
- SMITS, A. J., MCKEON, B. J. & MARUSIC, I. 2011 High-Reynolds number wall turbulence. *Annu. Rev. Fluid Mech.* **43**, 353–375.
- TALLURU, K. M., KULANDAIVELU, V., HUTCHINS, N. & MARUSIC, I. 2014 A calibration technique to correct sensor drift issues in hot-wire anemometry. *Meas. Sci. Technol.* **25** (10), 105304.
- VALLIKIVI, M., GANAPATHISUBRAMANI, B. & SMITS, A. J. 2015 Spectral scaling in boundary layers and pipes at very high Reynolds numbers. *J. Fluid Mech.* **771**, 303–326.